



**Grupo de Mecânica
Aplicada e Computacional**



**Federal University
of Santa Catarina**

The Element-Free Galerkin Method applied on Polymeric Foams

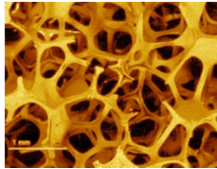
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Co-Advisor: Rodrigo Rossi, Dr.Eng.

Florianópolis - Brazil

September, 2006

Overview

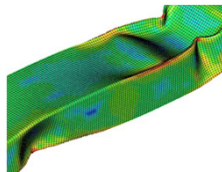
- **Celular Solids**



- Introduction
- Characteristics
- Mechanical Properties
- Models examples
- Methodology

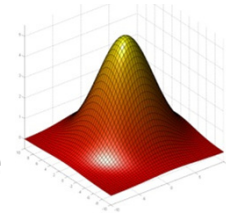
- **Finite Strain Elastoplastic formulation**

- Kinematics of deformation
- The concept of conjugated pairs
- The Elastoplastic model
- Strong, Weak and Incremental formulations
- Solution Procedure
- Examples (FEM)



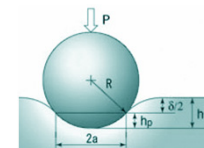
- **The Element-Free Galerkin Method**

- Moved List Square Approximation
- Weight Function definition
- Imposition of Essential Boundary Conditions

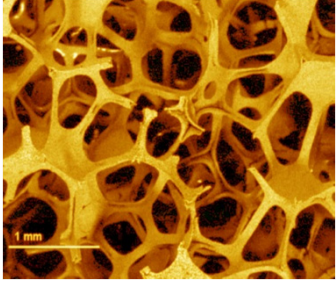


- **Unilateral Contact and Friction Formulation**

- Problem definition
- Imposition of the Contact and Friction terms
- General Algorithm
- Examples (EFG)



- **Conclusions**

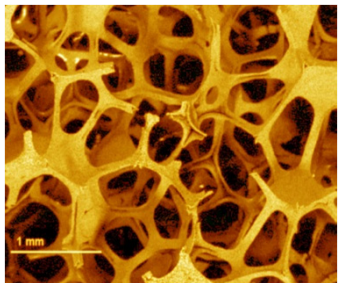


Cellular solids

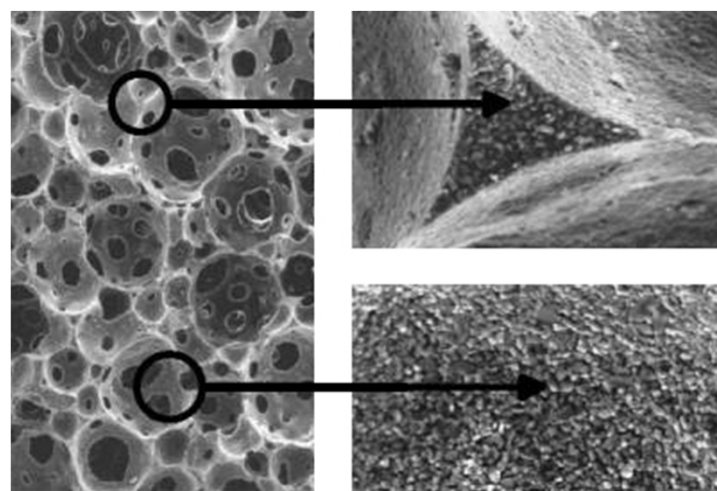
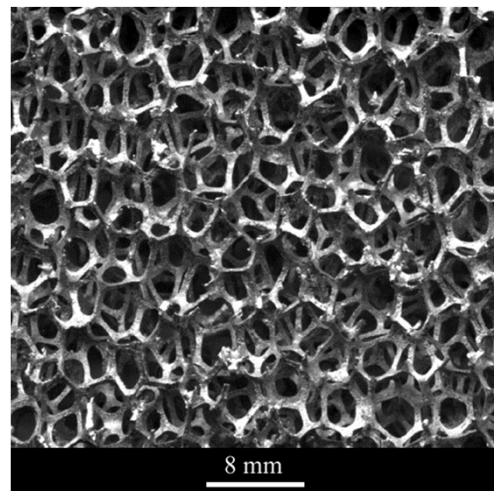
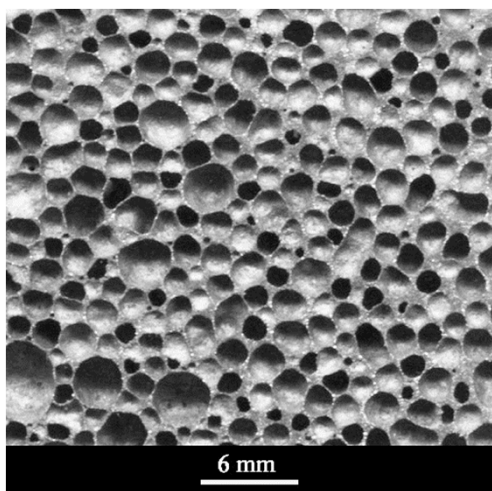
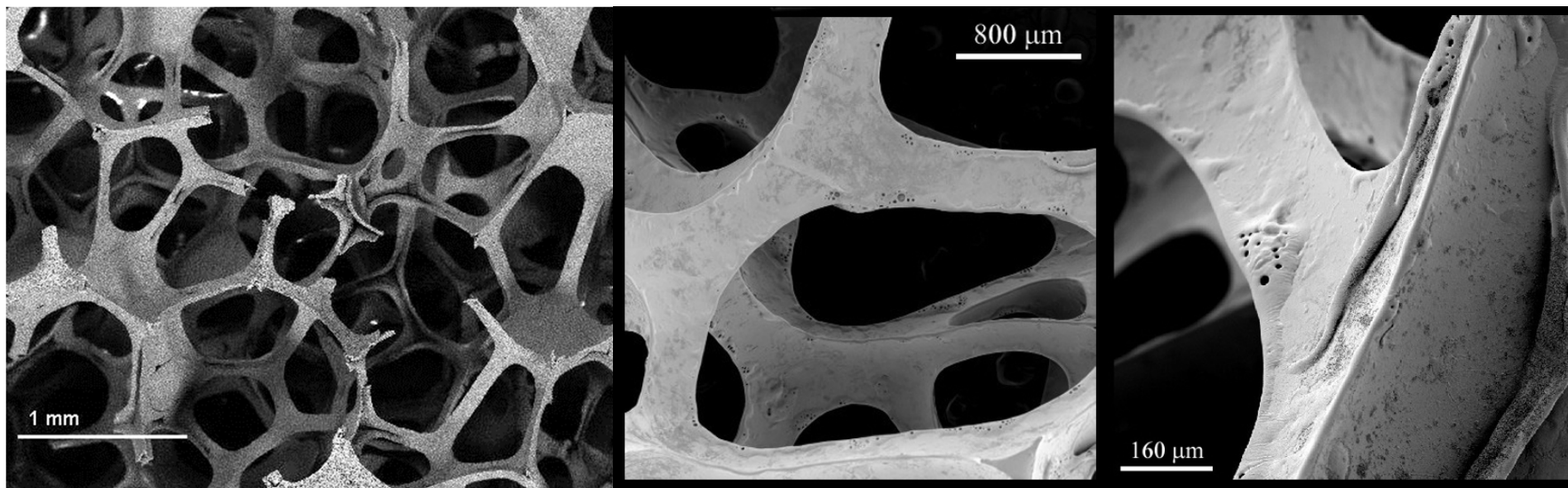
o Introduction

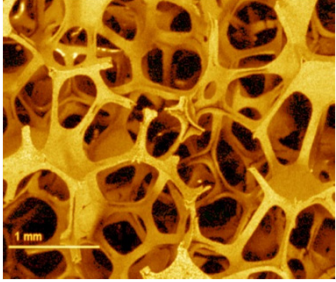
- Typically used in energy absorption structures.
- Applications areas:
 - Automotive/Transport industry;
 - Aerospace industry;
 - Packing industry;
 - Construction industry.
- Mechanical X low density.





Cellular solids





Cellular solids

o Relative density

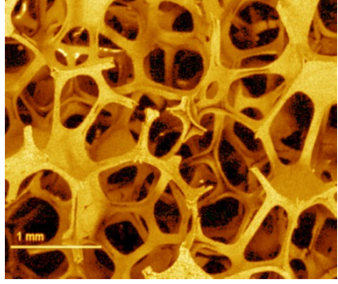
$$\rho^* = \rho/\rho_m$$

- Very low = 0,001;
- Conventional = 0,05 to 0,20;
- Transition value 0,3 treated as a solid with isolated pores.

o Types

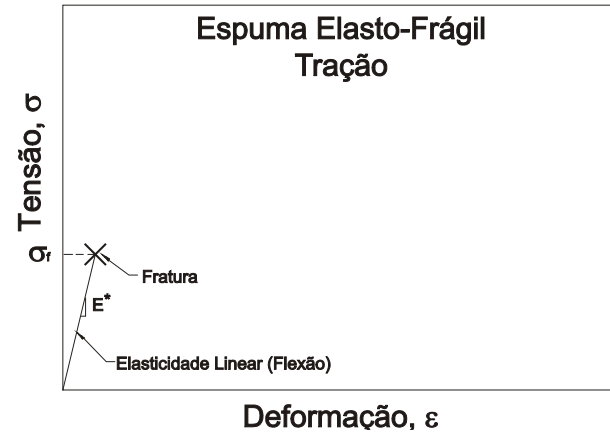
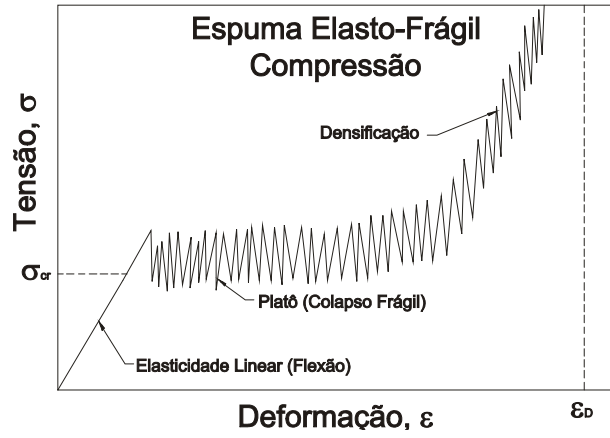
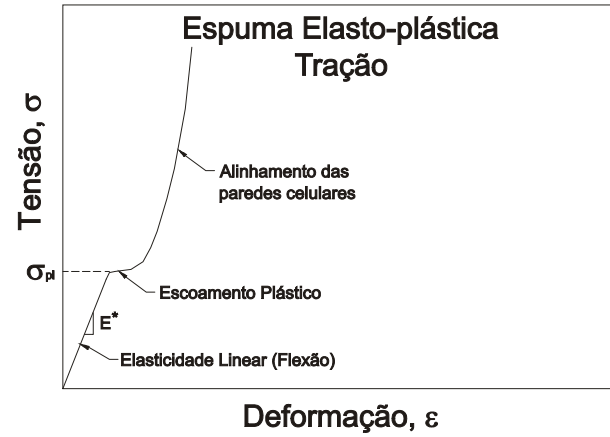
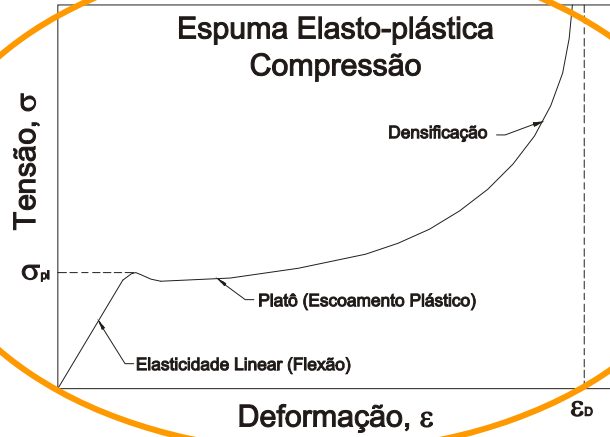
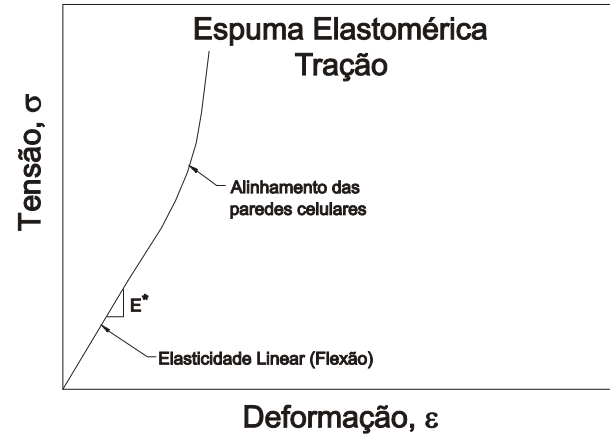
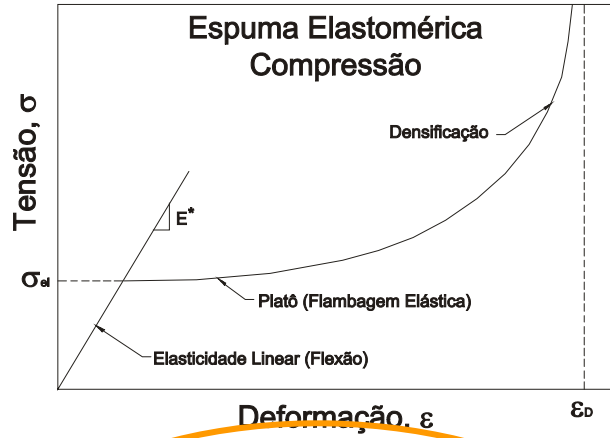
- Polymeric
- Metallic (aluminum, copper, nickel, titanium and zinc)
- Ceramic (carbon)
- Natural (wood, cork and coral structures)

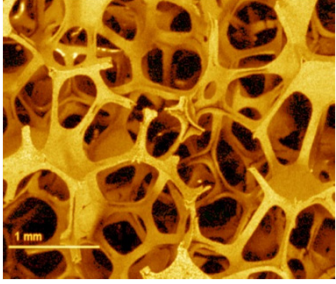




Mechanical Properties

GIBSON & ASHBY(1997)

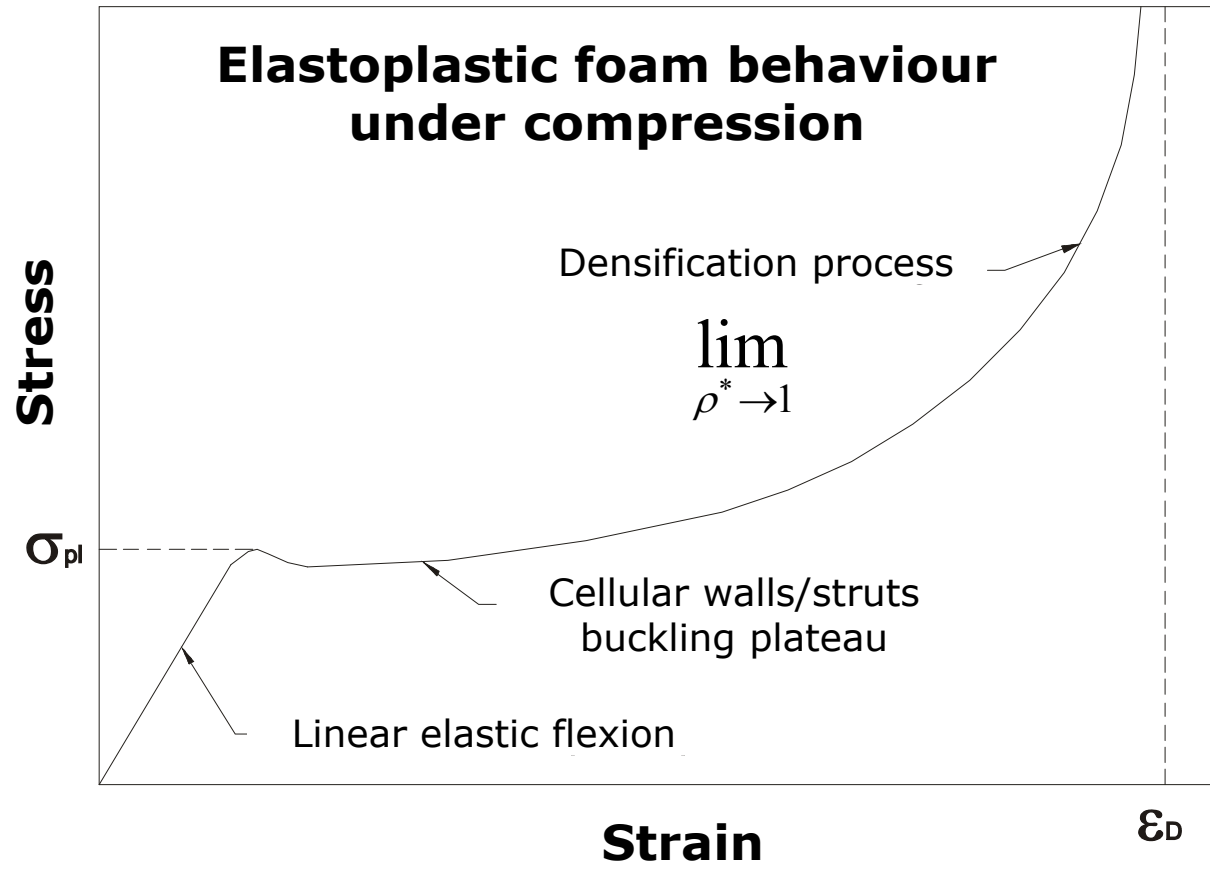


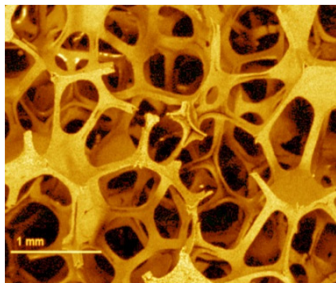


Cellular solids

- o **Mechanical Properties**

GIBSON & ASHBY(1997)



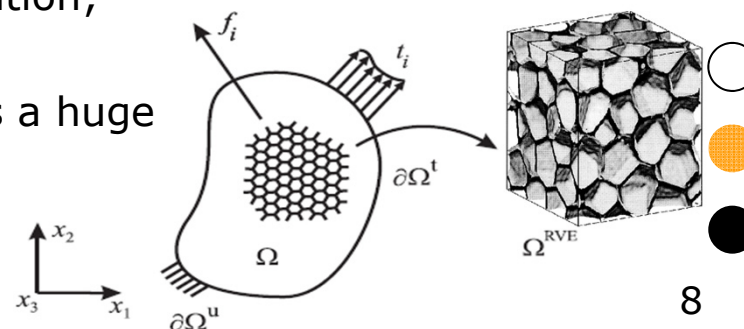
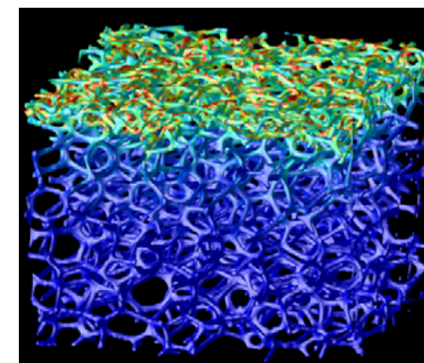
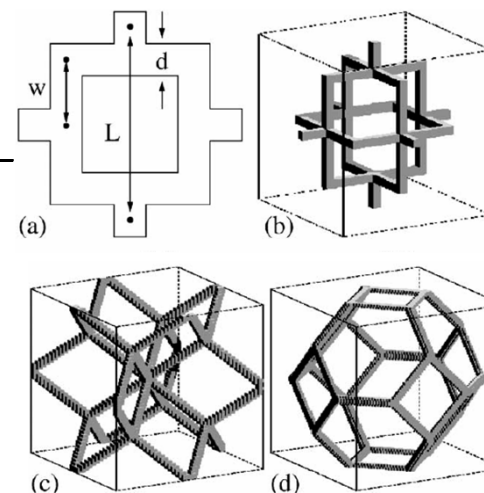


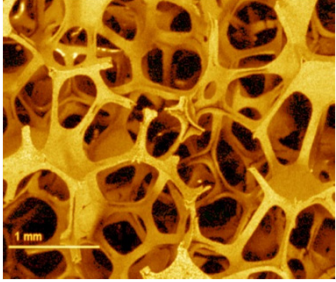
Cellular solids

o Modeling Approaches

- Periodical Models (GIBSON & ASHBY)
 - Dependence of the geometrical idealization;
 - Considers the local cell characteristics;
 - Limited applications;
 - Isotropy and cellular interaction, in the majority of the cases, are not periodic phenomenon.

- Random models (ROBERTS, GARBOCZIA e BRYDON)
 - *Voronoi* tessellation and *Gaussian* random field;
 - Problems to define the RVE and the coordinate number;
 - Problems at the micro tomography correlation;
 - Border effects;
 - Self-contact densification process involves a huge computational cost;
 - Explicit solver.





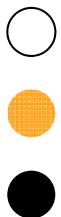
Cellular solids

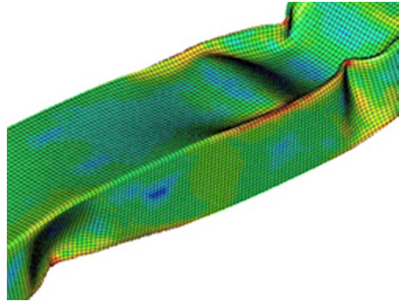
- **Elastoplastic model for polymeric foams**

- **Methodology**
 - Finite Stains Algorithm;
 - *Total Lagrange* Description;
 - *Rotated Kirchoff* stress;
 - *Hencky* Logarithmic strain measure;
 - Volumetric hardening Law;

 - Finite Element Method - FEM;
 - Element Free Galerkin Method - EFG;

 - Contact formulation (*Signorini* hypothesis);
 - Friction formulation (regularized *Coulomb's* law);



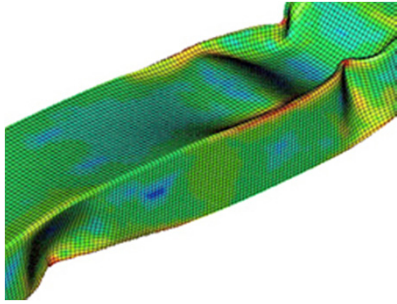


Finite Strain Elastoplastic Formulation

○ Required formulations

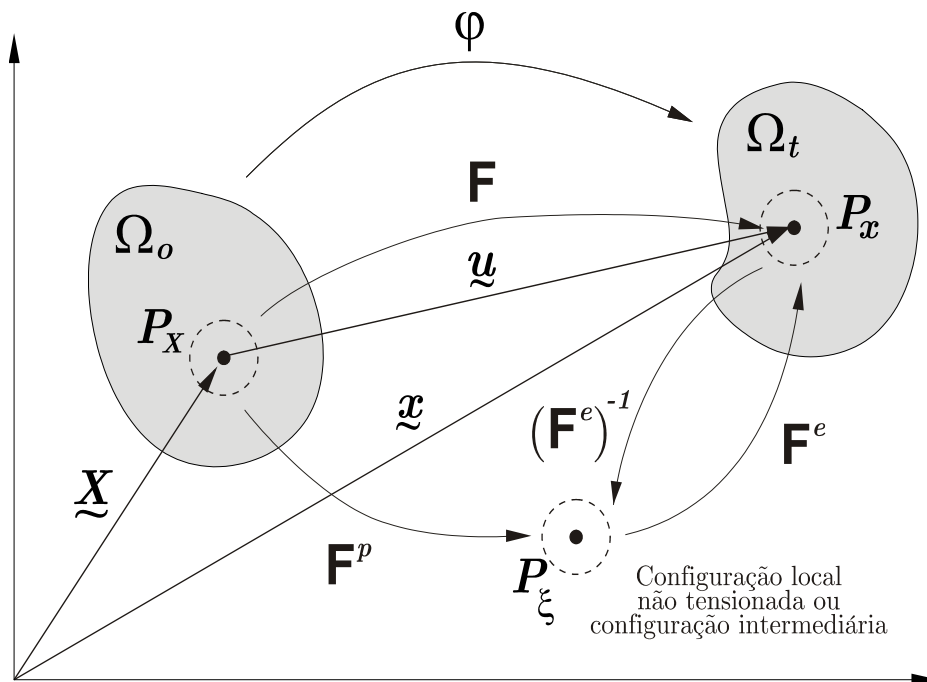
- Stress/Strain elastic relation – an elastic law;
- Yield function - indicating the stress level to start the plastic flow;
- Stress/Strain plastic relation – material hardening/softening law;





Finite Strain Elastoplastic Formulation

o Kinematics of deformation



- Movement, strain gradient and the multiplicative decomposition

$$\varphi(\vec{X}, t) = \vec{x} = \vec{X} + \vec{u}$$

$$\mathbf{F} = \nabla_{\vec{X}} \varphi(\vec{X}, t) = \frac{\partial \vec{x}}{\partial \vec{X}}$$

$$\mathbf{F} = \mathbf{F}^e \mathbf{F}^p$$

- Polar decomposition, *Cauchy-Green* tensor and the log strain measure

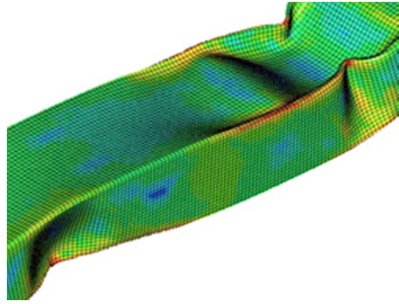
$$\mathbf{F}^p = \mathbf{R}^p \mathbf{U}^p \quad \mathbf{F}^e = \mathbf{R}^e \mathbf{U}^e$$

$$\mathbf{U}^e = \sqrt{\mathbf{C}^e}$$

$$\mathbf{C}^e = \mathbf{F}^{eT} \mathbf{F}^e$$

$$\mathbf{E}^e = \ln(\mathbf{U}^e)$$





Finite Strain Elastoplastic Formulation

- **Hill's Principle: work rate invariability**

$$\dot{\mathcal{W}} = \frac{1}{\rho} \boldsymbol{\sigma} \cdot \mathbf{D} = \frac{1}{\rho_0} \boldsymbol{\tau} \cdot \mathbf{D} = \frac{1}{\rho_0} \mathbf{P} \cdot \dot{\mathbf{F}} = \frac{1}{2\rho_0} \mathbf{S} \cdot \dot{\mathbf{C}} = \frac{1}{\rho_0} \bar{\boldsymbol{\tau}} \cdot \dot{\mathbf{E}}^e,$$

- **Spectral decomposition**

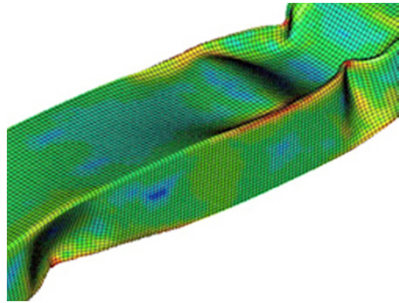
$$\mathbf{C}^e = \mathbf{F}^{eT} \mathbf{F}^e$$

$$\mathbf{C}^e = \sum_{i=1}^3 \lambda_i (\vec{l}_i \otimes \vec{l}_i)$$

$$\mathbf{U}^e = \sum_{i=1}^3 \sqrt{\lambda_i} (\vec{l}_i \otimes \vec{l}_i)$$

$$\mathbf{E}^e = \frac{1}{2} \sum_{i=1}^3 \ln(\lambda_i) (\vec{l}_i \otimes \vec{l}_i)$$





Finite Strain Elastoplastic Formulation

- **Kirchoff rotated stress**

$$\bar{\boldsymbol{\tau}} = (\mathbf{R}^e)^T \boldsymbol{\tau} (\mathbf{R}^e) \quad \text{where} \quad \boldsymbol{\tau} = \det(\mathbf{F}) \boldsymbol{\sigma}$$

- **Hyperelastic Hencky's model**

$$\bar{\boldsymbol{\tau}} = \mathbb{D}(\rho^*) \mathbf{E}^e$$

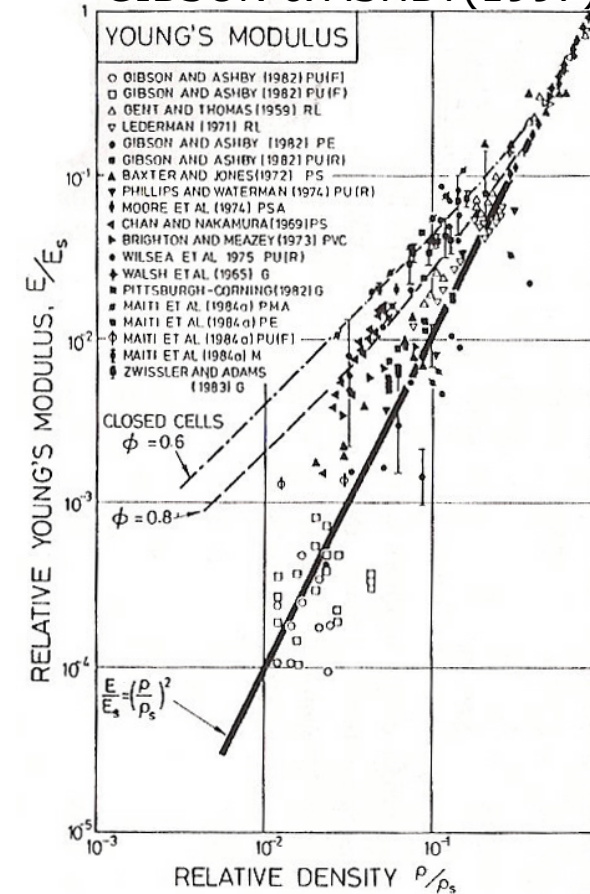
$$\mathbb{D}(\rho^*) = 2\mu(\rho^*) \mathbb{I} + \left(K(\rho^*) - \frac{2}{3} \mu(\rho^*) \right) (\mathbf{I} \otimes \mathbf{I})$$

$$\rho_o^* = \det[\mathbf{F}] \rho^*$$

$$\mathbb{I}_{ijkl} = \frac{1}{2} (\delta_{ik} \delta_{jl} + \delta_{il} \delta_{jk})$$

$$(\mathbf{I} \otimes \mathbf{I})_{ijkl} = \delta_{ij} \delta_{kl}$$

GIBSON & ASHBY(1997)



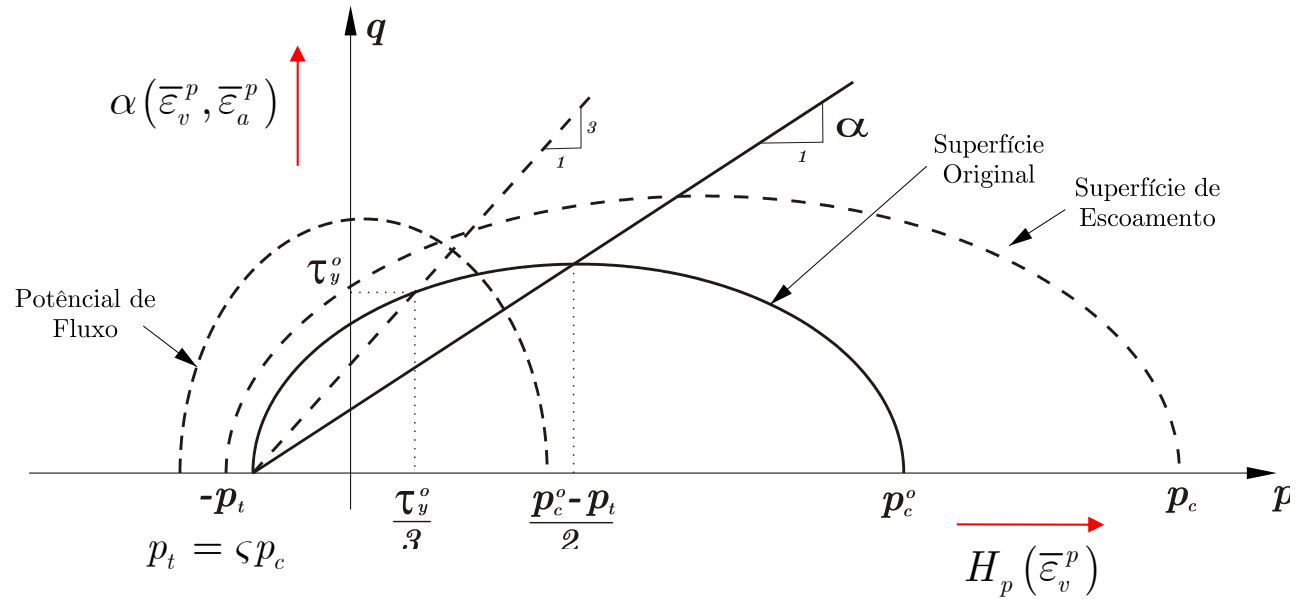
$$E(\rho^*) = c(\rho^*)^\gamma E_M$$



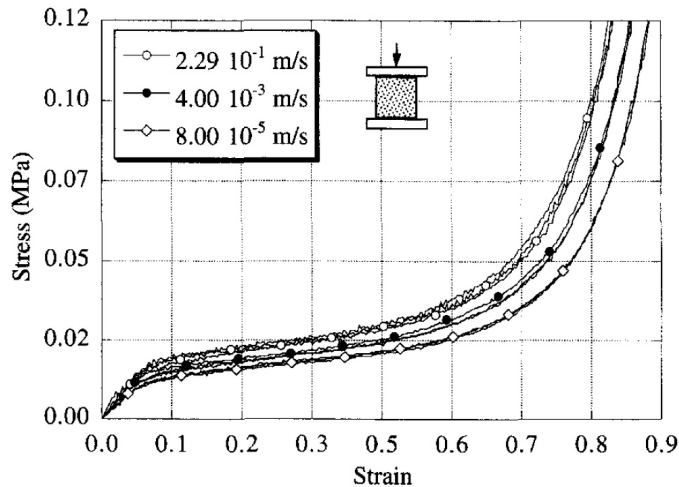
Yield Function and the Plastic Flow Potential

$$\mathcal{F}(q, p) = \sqrt{q^2 + \alpha^2 \left(p - \left[\frac{p_c - p_t^o}{2} \right] \right)^2} - \alpha \left[\frac{p_c + p_t^o}{2} \right] \leq 0$$

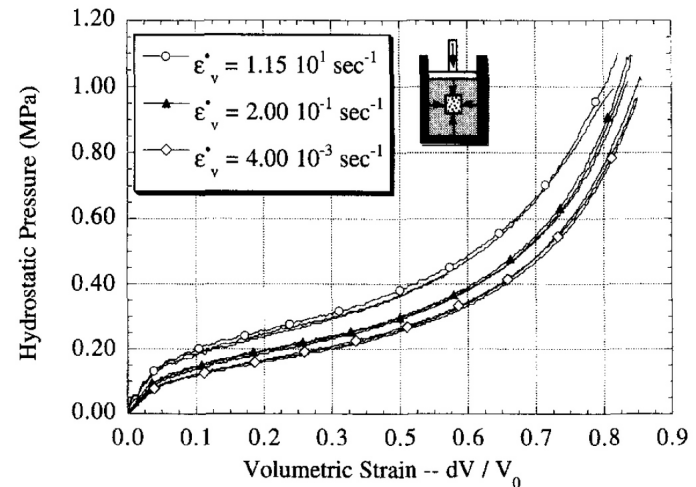
$$\mathcal{G}(q, p) = \sqrt{q^2 + \beta_{(\nu_p)}^2 p^2}$$



$$\bar{\tau}_y(\bar{\epsilon}_a^p) = \bar{\tau}_y^o + H(\bar{\epsilon}_a^p) \quad \bar{\epsilon}_a^p = -\ln\left(\frac{L^p}{L_o}\right)$$



$$p_c = p_c^o + H_p(\bar{\epsilon}_v^p) \quad \bar{\epsilon}_v^p = -\ln(J^p)$$



Elastic Prediction

$$\dot{\mathbf{F}}^p = \mathbf{0}$$

$$\mathbf{F}_{n+1}^{p\text{ teste}} = \mathbf{F}_n^p$$

$$\mathbf{F}_{n+1}^{e\text{ teste}} = \mathbf{F}_{n+1} (\mathbf{F}_n^p)^{-1}$$

$$\rho_{n+1}^* = \frac{\rho_o^*}{\det[\mathbf{F}_{n+1}]}$$

$$\mathbf{C}_{n+1}^{e\text{ teste}} = (\mathbf{F}_{n+1}^{e\text{ teste}})^T \mathbf{F}_{n+1}^{e\text{ teste}}$$

$$\mathbf{E}_{n+1}^{e\text{ teste}} = \frac{1}{2} \ln(\mathbf{C}_{n+1}^{e\text{ teste}})$$

$$\bar{\boldsymbol{\tau}} = \mathbb{D}(\rho_{n+1}^*) \mathbf{E}_{n+1}^{e\text{ teste}}$$

$$p_{n+1}^{\text{ teste}} = -K(\rho_{n+1}^*) \text{tr}[\mathbf{E}_{n+1}^{e\text{ teste}}]$$

$$q_{n+1}^{\text{ teste}} = \sqrt{\frac{3}{2} \bar{\boldsymbol{\tau}}_{n+1}^{D\text{ teste}} \cdot \bar{\boldsymbol{\tau}}_{n+1}^{D\text{ teste}}}$$

Return Mapping Algorithm

End

$$\mathbf{F}_{n+1}^p = \exp\left(\Delta\lambda \frac{\partial \mathcal{G}}{\partial \bar{\boldsymbol{\tau}}}\bigg|_{n+1}\right) \mathbf{F}_n^p$$

$$\begin{cases} \left[1 + \frac{\kappa\beta^2\Delta\lambda}{\mathcal{G}(p_{n+1}, q_{n+1})}\right] p_{n+1} - p_{n+1}^{\text{ teste}} \\ \left[1 + \frac{3\mu\Delta\lambda}{\mathcal{G}(p_{n+1}, q_{n+1})}\right] q_{n+1} - q_{n+1}^{\text{ teste}} = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} \\ \mathcal{F}(q_{n+1}, p_{n+1}, \Delta\lambda) \end{cases}$$

in terms of $p_{n+1}^{\text{ teste}}$ $q_{n+1}^{\text{ teste}}$ $\Delta\lambda$

where $\Delta\lambda > 0$

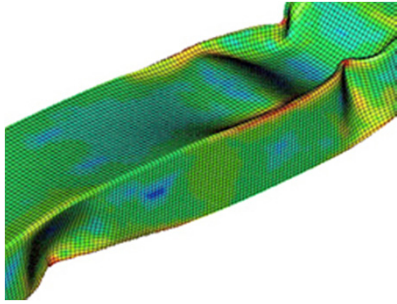
Plastic Corrector

$$\mathcal{F}(q_{n+1}^{\text{ teste}}, p_{n+1}^{\text{ teste}}, \varepsilon_{v_{n+1}}^{\text{ teste}}) \leq 0$$

no

yes

$$(\cdot)_{n+1} = (\cdot)_{n+1}^{\text{ teste}}$$

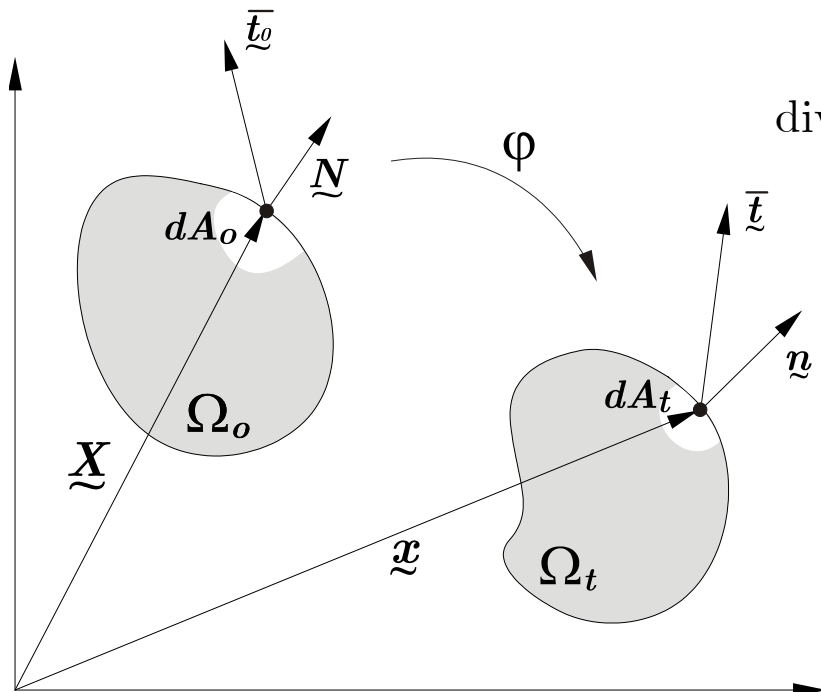


Finite Strain Elastoplastic Formulation

- o **The global problem value**

- **Strong formulation: reference configuration**

- For each $t \in [t_o, t_f]$, determine $\vec{u}(\vec{X}, t)$ that it is solution of

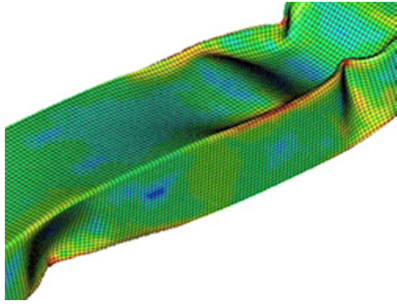


$$\text{div } \mathbf{P}(\vec{X}, t) - \rho_o(\vec{X}) \vec{b}(\vec{X}, t) = 0 \quad \text{em } \Omega_o$$

$$\mathbf{P}(\vec{X}, t) \vec{N}(\vec{X}, t) = \vec{t}(\vec{X}, t) \quad \text{em } \Gamma_o^t$$

$$\vec{u}(\vec{X}, t) = \vec{u}(\vec{X}) \quad \text{em } \Gamma_o^u$$





Finite Strain Elastoplastic Formulation

- **The global problem value**
 - **Weak formulation: reference configuration**

- Find $\vec{u}_{n+1} \in \mathcal{K}$ such that

$$\mathcal{F}(\vec{u}_{n+1}; \delta\vec{u}) = 0 \quad \forall \delta\vec{u} \in \mathcal{V}$$

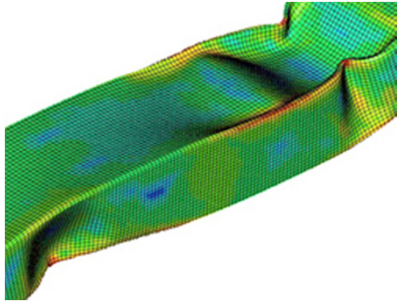
where

$$\mathcal{F}(\vec{u}_{n+1}; \delta\vec{u}) = \int_{\Omega_o} \mathbf{P}(\vec{u}_{n+1}) \cdot \nabla_{\vec{X}} \delta\vec{u} \, d\Omega_o - \int_{\Omega_o} \rho_o \vec{b}_{n+1} \cdot \delta\vec{u} \, d\Omega_o - \int_{\Gamma_o^t} \vec{t}_{n+1} \cdot \delta\vec{u} \, dA_o$$

$$\mathcal{K} = \left\{ \vec{u} \mid u_i \in W_p^1(\Omega), \vec{u} = \vec{\bar{u}} \text{ em } \Gamma_o^u \right\}$$

$$\mathcal{V} = \left\{ \delta\vec{u} \mid \delta u_i \in W_p^1(\Omega), \delta\vec{u} = 0 \text{ em } \Gamma_o^u \right\}$$





Finite Strain Elastoplastic Formulation

o Local Linearization (*Newton's Method*)

- Considers $F(\cdot, \cdot)$ as being enough regular

$$F\left(\vec{u}_{n+1}^{k+1}; \delta\vec{u}\right) = F\left(\vec{u}_{n+1}^k + \Delta\vec{u}_{n+1}^k; \delta\vec{u}\right) = 0 \quad \forall \delta\vec{u} \in \mathcal{V}$$

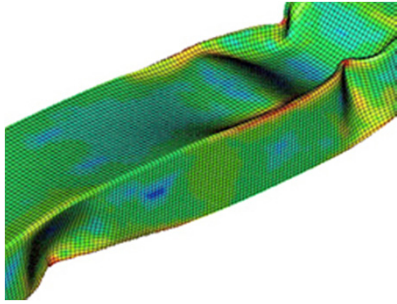
and expanding by *Taylor* in terms of \vec{u}_{n+1}^k , we obtain

$$F\left(\vec{u}_{n+1}^k + \Delta\vec{u}_{n+1}^k; \delta\vec{u}\right) \simeq F\left(\vec{u}_{n+1}^k; \delta\vec{u}\right) + DF\left(\vec{u}_{n+1}^k; \delta\vec{u}\right)\left[\Delta\vec{u}_{n+1}^k\right]$$

$$DF\left(\vec{u}_{n+1}^k; \delta\vec{u}\right)\left[\Delta\vec{u}_{n+1}^k\right] = \int_{\Omega_o} \mathbb{A}\left(\vec{u}_{n+1}^k\right) \cdot \nabla_{\vec{X}}\left(\Delta\vec{u}_{n+1}^k\right) \cdot \nabla_{\vec{X}}\delta\vec{u} \, d\Omega_o$$

$$\left[\mathbb{A}\left(\vec{u}_{n+1}^k\right)\right]_{ijkl} = \left.\frac{\partial P_{ij}}{\partial F_{kl}}\right|_{\vec{u}_{n+1}^k} = \frac{\partial \tau_{ip}}{\partial F_{kl}} F_{jp}^{-1} - \tau_{ip} F_{jk}^{-1} F_{lp}^{-1}$$





Finite Strain Elastoplastic Formulation

o Global Equilibrium

- Find \vec{u}_{n+1}^h , that satisfies the residual equilibrium equation

$$\vec{r}(\vec{u}_{n+1}^h) = \vec{f}^{\text{int}}(\vec{u}_{n+1}^h) - \vec{f}_{n+1}^{\text{ext}} = 0$$

$$\vec{f}^{\text{int}}(\vec{u}_{n+1}^h) = \int_{\Omega_o} (\mathbf{G}^g)^T \vec{P}(\vec{u}_{n+1}^h) d\Omega_o \quad \vec{f}_{n+1}^{\text{ext}} = \int_{\Omega_o} \rho_o (\Phi^g)^T \vec{b}_o d\Omega_o + \int_{\Gamma_o^t} (\Phi^g)^T \vec{t}_{o_{n+1}} dA_o$$

o Global Linearization (Newton's Method)

$$\vec{r}(\vec{u}_{n+1}^{h^{k+1}}) = \vec{r}(\vec{u}_{n+1}^{h^k} + \Delta \vec{u}_{n+1}^{h^k}) = \vec{0}$$

$$\vec{r}(\vec{u}_{n+1}^{h^k} + \Delta \vec{u}_{n+1}^{h^k}) \simeq \vec{r}(\vec{u}_{n+1}^{h^k}) + D\vec{r}(\vec{u}_{n+1}^{h^k})[\Delta \vec{u}_{n+1}^{h^k}]$$

$$D\vec{r}(\vec{u}_{n+1}^{h^k})[\Delta \vec{u}_{n+1}^{h^k}] = \mathbf{K} \Delta \vec{u}_{n+1}^{g^k}$$

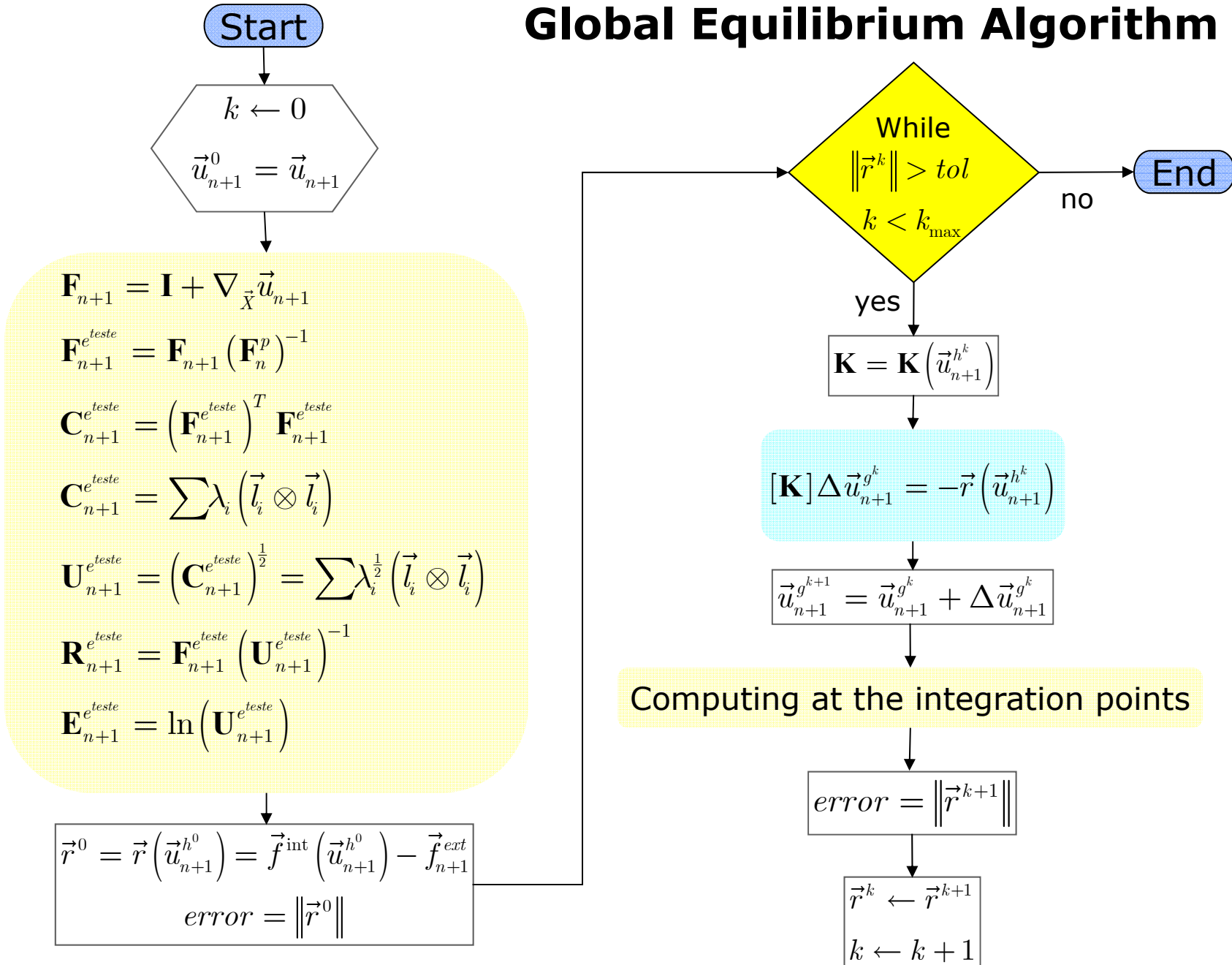
$$\mathbf{K} = \int_{\Omega_o} (\mathbf{G}^g)^T \mathbf{A} \mathbf{G}^g d\Omega_o$$

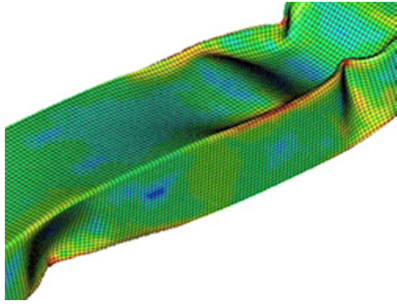
$$[\mathbf{K}] \Delta \vec{u}_{n+1}^{g^k} = -\vec{r}(\vec{u}_{n+1}^{h^k})$$

for $\Delta \vec{u}_{n+1}^{g^k}$



Global Equilibrium Algorithm

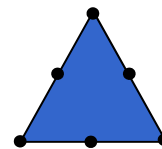
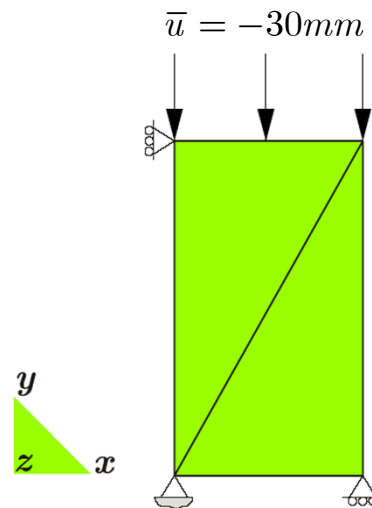
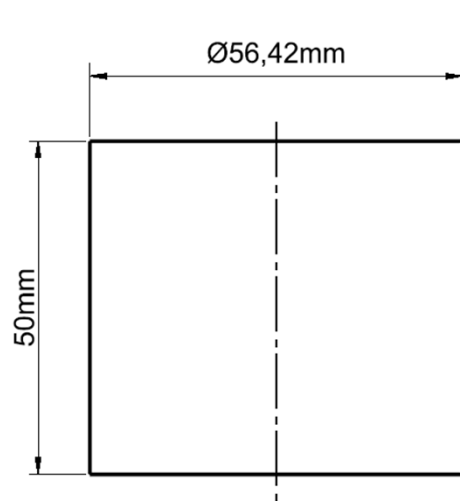




Finite Strain Elastoplastic Formulation

Examples

- Axisymmetric Uniaxial Compression



TRI6

$$\tau_y^0 = 0,082034 \text{ MPa}$$

$$p_c^0 = 0,040470 \text{ MPa}$$

$$\zeta = 0,10$$

$$E_m = 928,092 \text{ MPa}$$

$$\rho_0^* = 0,049 \text{ Kg} / \text{m}^3$$

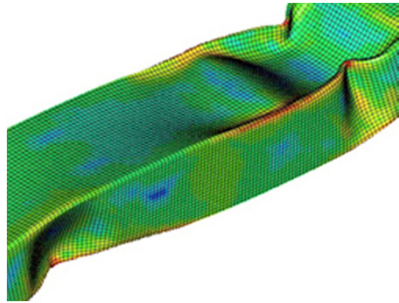
$$\nu_p = 0,00$$

$$\nu = 0,25$$

$$\gamma = 1,54$$

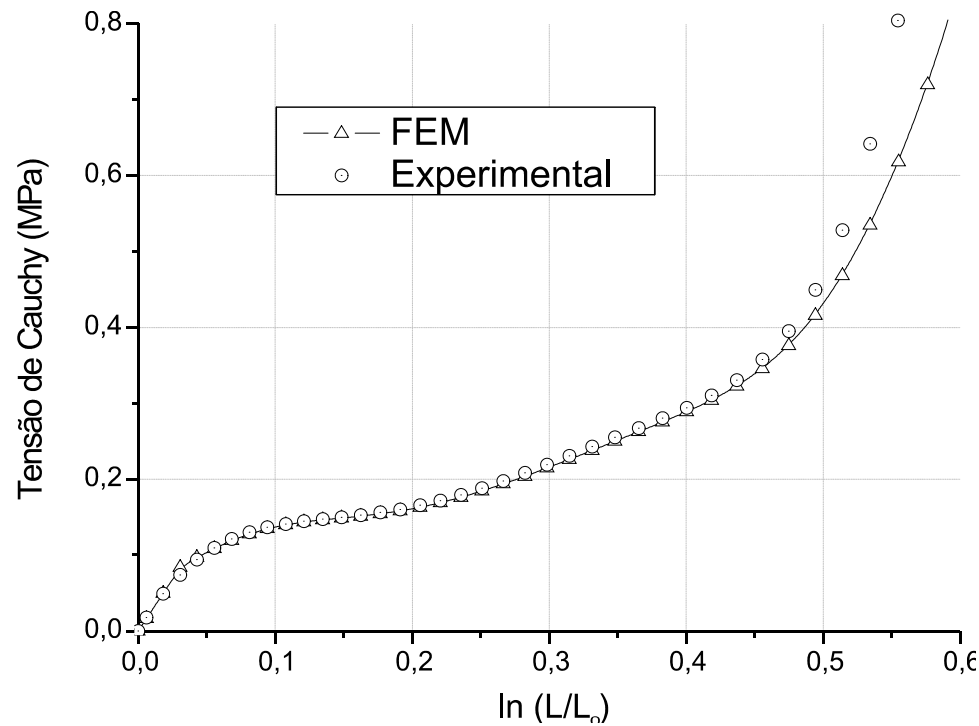
$$c = 0,30$$



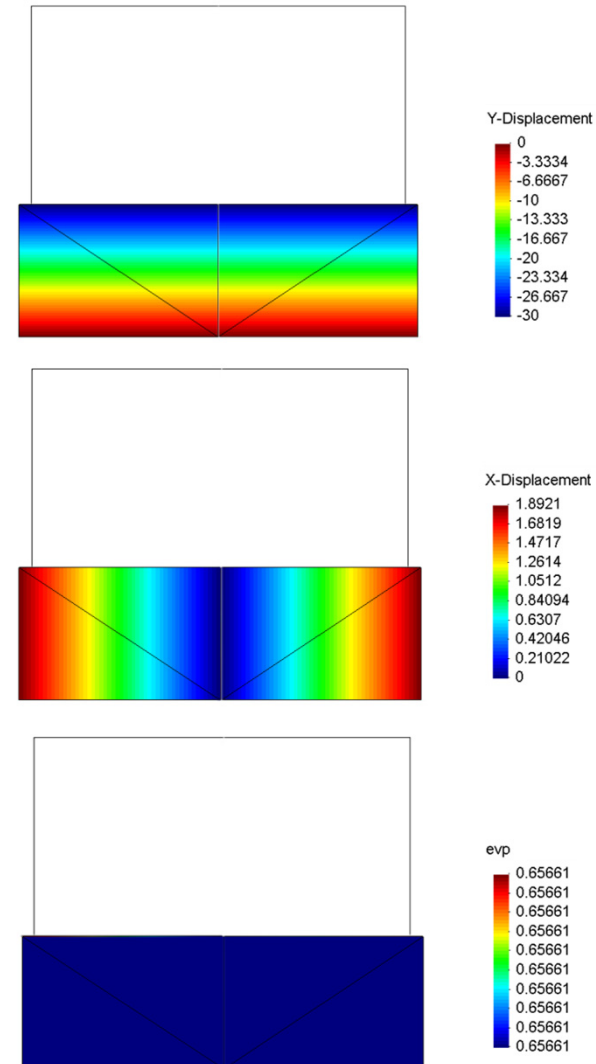


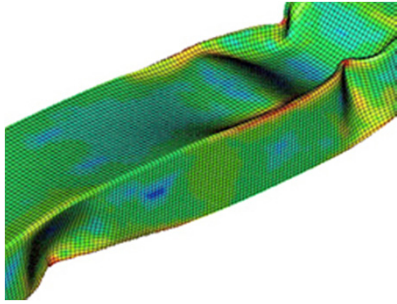
Finite Strain Elastoplastic Formulation

- Uniaxial Compression (Axisymmetric)



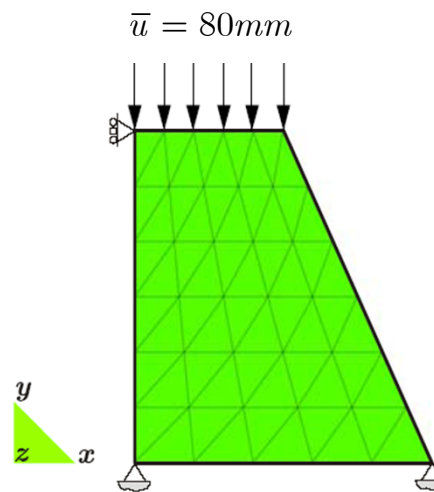
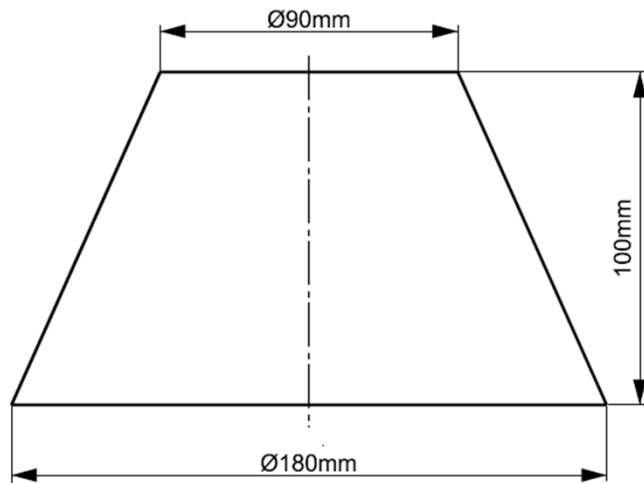
Experimental data ZHANG, J. *at al* (1998)



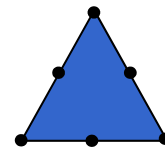


Finite Strain Elastoplastic Formulation

- Multiaxial Compression – Axisymmetric Cone Slice

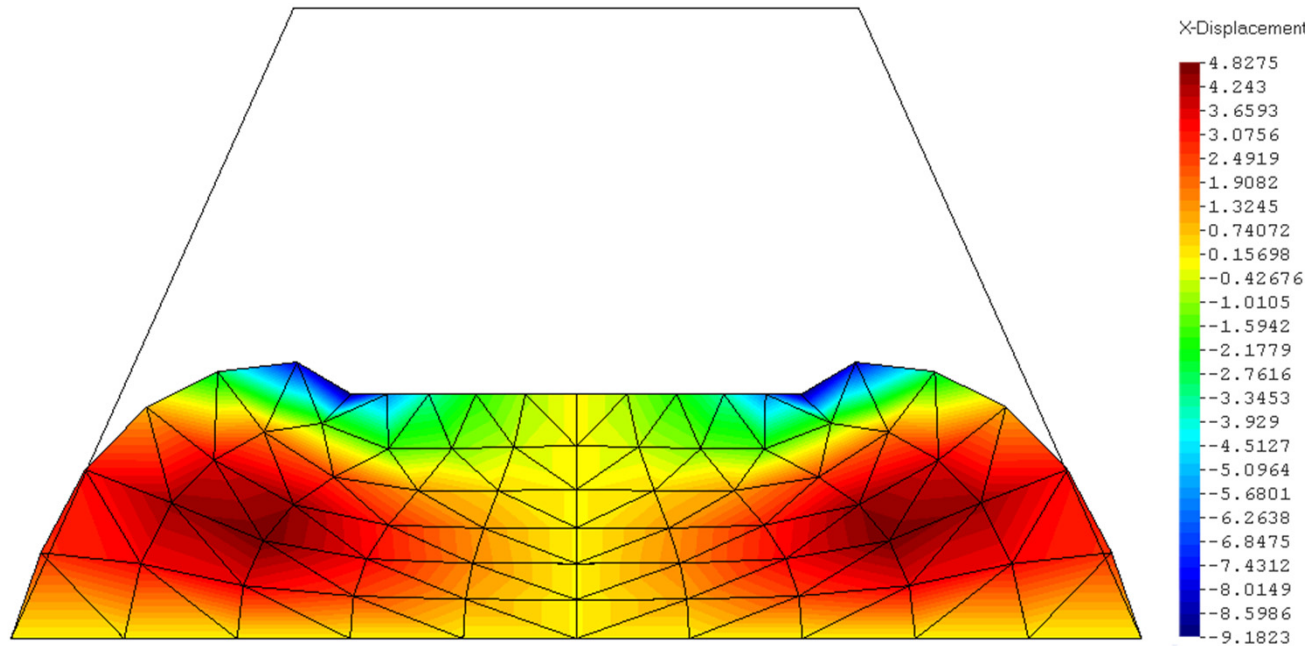


$$\begin{aligned} \tau_y^o &= 0,082 \text{ MPa} \\ p_c^o &= 0,040 \text{ MPa} \\ \zeta &= 0,01 \\ E_m &= 928,092 \text{ MPa} \\ \rho_o^* &= 0,049 \text{ Kg} / \text{m}^3 \\ \nu_p &= 0,00 \\ \nu &= 0,25 \\ \gamma &= 1,54 \\ c &= 0,30 \end{aligned}$$

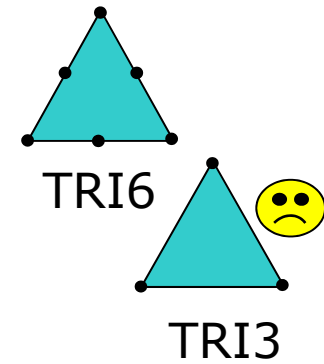
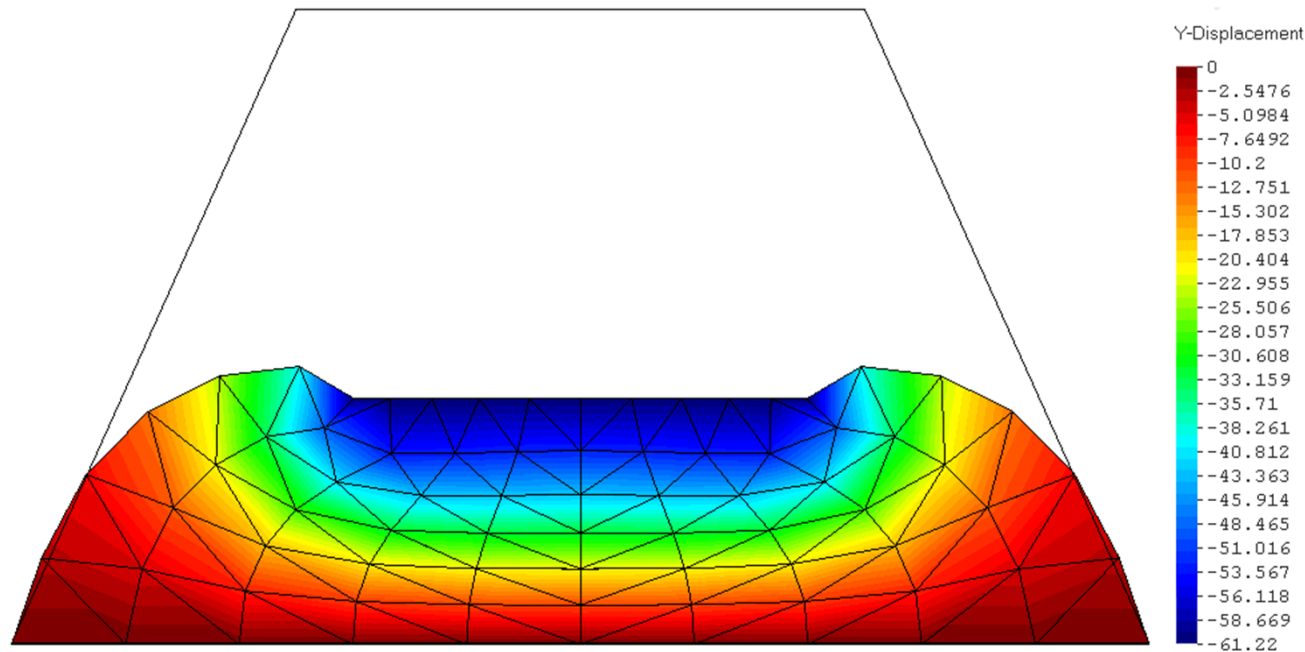


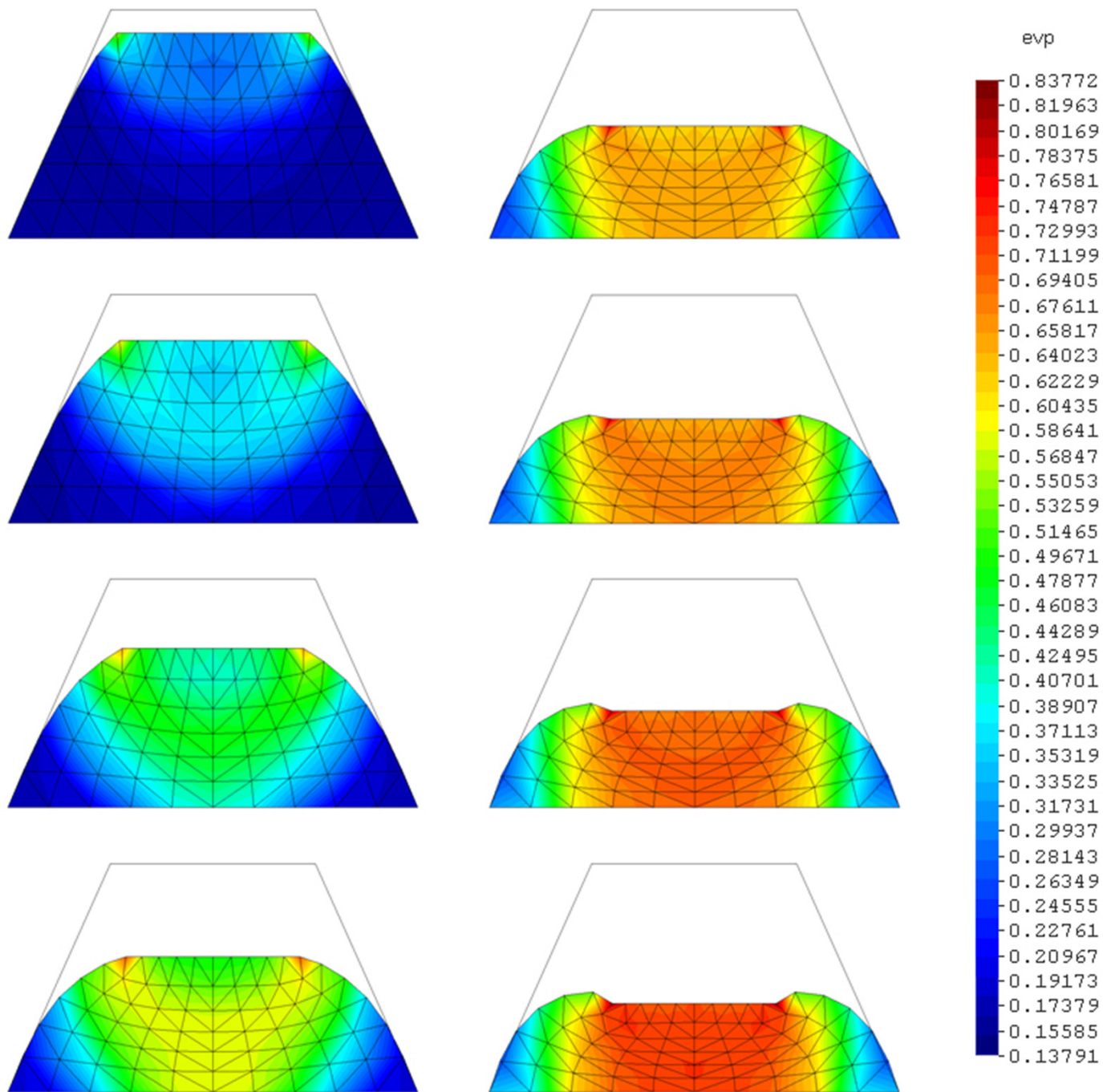
TRI6

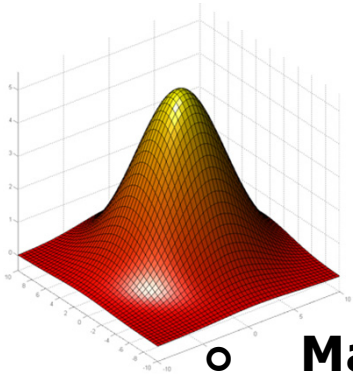




Note: Calculated with TRI6 visualized as TRI3 element!







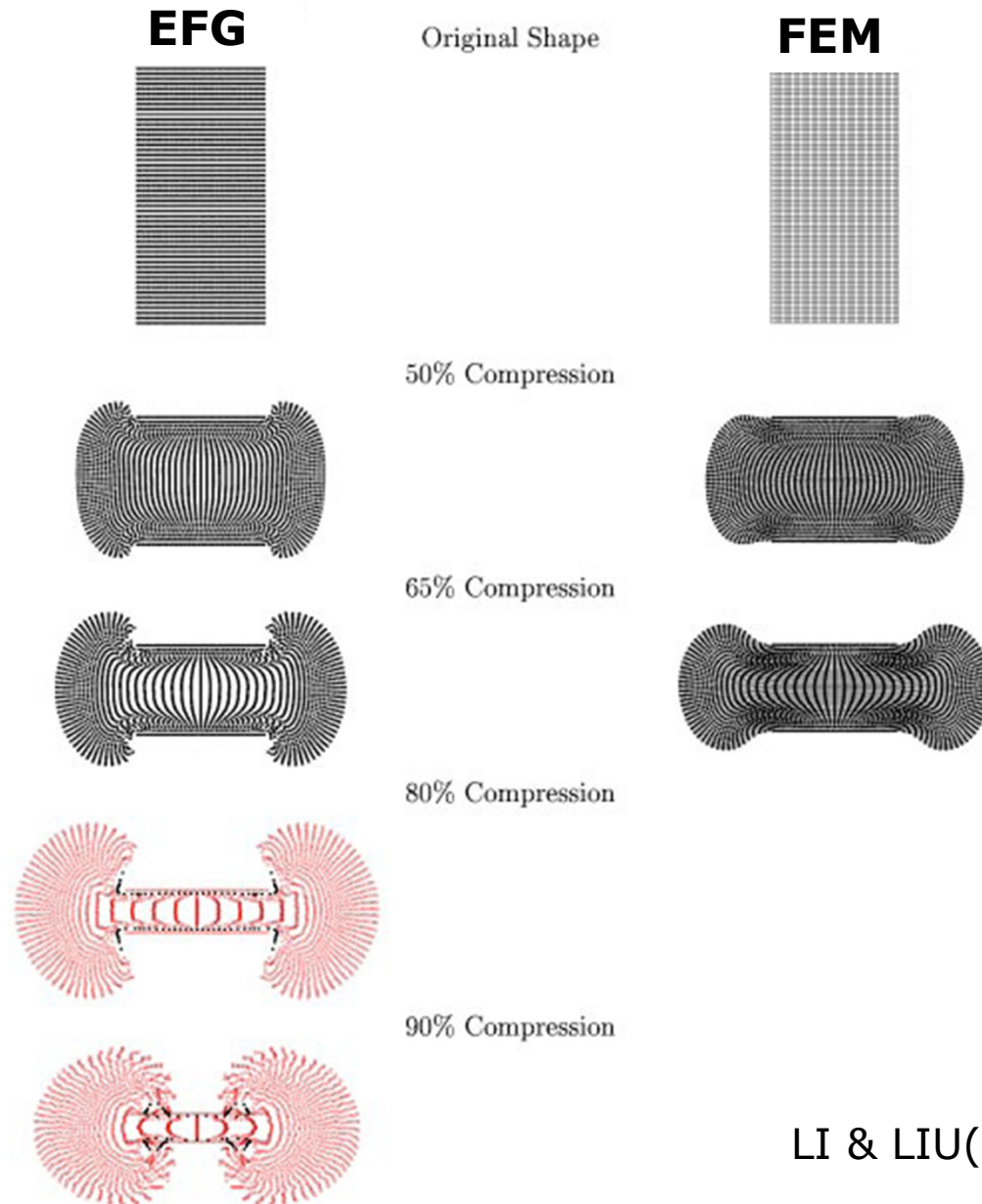
Element-Free Galerkin Method (EFG)

o **Main characteristics** BELYTSCHKO *et al.* (1994)

- The discretization of the differential equations system uses the weak Galerkin form;
- Do not required a finite element mesh;
- The discretization is based on a set of nodes (discrete data);
- The connection in terms of nodes interactions may be change constantly, and modeling fracture (ability to simulate crack growth), free surfaces, large deformations, etc. is considerably simplified;
- Accuracy can be controlled more easily, since in areas where more refinement is needed, nodes can be added easily;
- Meshfree discretization can provide accurate representation of geometric object;
- A special strategy to impose the essential boundary conditions.

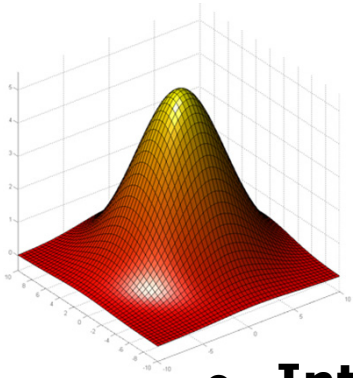


Comparison of the deformations at different time stages for a block of hyperelastic material under compression by using:



LI & LIU(2002)





Element-Free Galerkin Method (EFG)

o Introduction

- Enables the construction of an approximate function

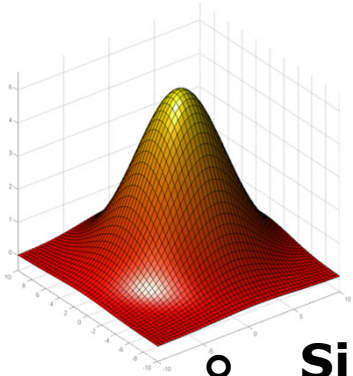
$$u^h(\mathbf{X})$$

that fits a discrete set of data

$$u_I = \{u_I, I=1, \dots, n\}$$

- *Moving Least Square Approximation (MLSA).*
LANCASTER & SALKAUSKAS (1981);





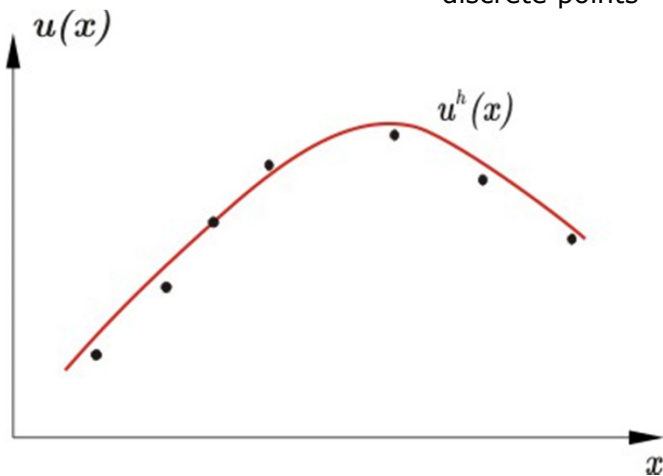
Element-Free Galerkin Method (EFG)

o Simple Least Squares Method

- Finding the best-fitting curve to a given set of discrete points (or nodes)

$$u_I = \{u_I, I=1, \dots, n\} \quad \text{discrete points} \quad \Rightarrow \quad u^h(\vec{x}) = \sum_{j=1}^m p_j(\vec{x}) a_j = \langle \vec{p}(\vec{x}), \vec{a} \rangle$$

discrete points approximation function



$$J(\vec{a}^*) = \sum_{I=1}^n \{u^h(x_I) - u_I\}^2 = \sum_{I=1}^n \left\{ \langle \vec{p}, \vec{a}^* \rangle - u_I \right\}^2$$

$$\vec{a} = \arg \min \{J(\vec{a}^*)\} \quad \forall \vec{a}^* \in \mathcal{R}^m$$

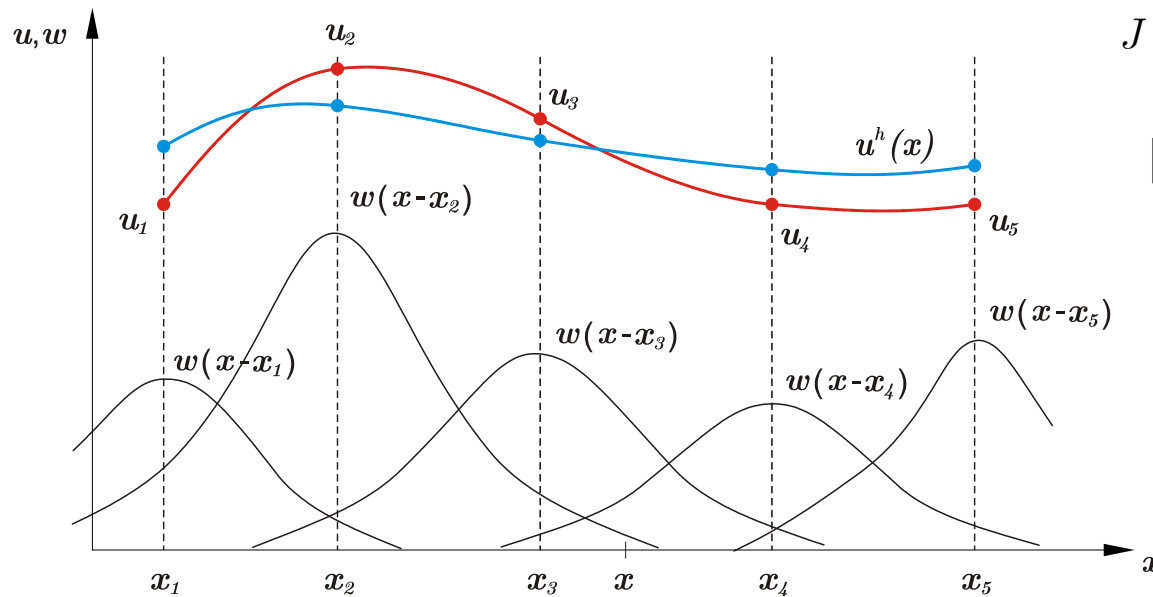
$$[\mathbf{A}] \vec{a} = \sum_{I=1}^n u_I \vec{b}_I$$

where $[\mathbf{A}] = \sum_{I=1}^n [\vec{p}(x_I) \otimes \vec{p}(x_I)]$ and $\vec{b}_I = \vec{p}(x_I)$.

- **ALL** values $u_I, I=1, \dots, n$ exerts influence to determinate **ALL** the a_j coefficients, for the n discrete points.



The Moving Least Square Approximation (MLSA)



$$J(\vec{a}) = \sum_{I=1}^n w(\vec{x} - \vec{x}_I) \left[\langle \vec{p}(\vec{x}_I), \vec{a}(\vec{x}) \rangle - u_I \right]^2$$

$$[\mathbf{A}(x)] = \sum_{I=1}^n w(x - x_I) [\vec{p}(x_I) \otimes \vec{p}(x_I)]$$

$$\vec{b}_I = w(x - x_I) \vec{p}(x_I)$$

$$[\mathbf{A}(x)] \vec{a}(x) = \sum_{I=1}^n u_I \vec{b}_I$$

$$\vec{a}(x) = [\mathbf{A}(x)]^{-1} \left\{ \sum_{I=1}^n u_I \vec{b}_I \right\}$$

$$u^h(x) = \langle \vec{p}(x), \vec{a}(x) \rangle = \sum_{I=1}^n u_I \langle \vec{p}(x), [\mathbf{A}(x)]^{-1} \vec{b}_I \rangle = \sum_{I=1}^n u_I \Phi_I(x)$$

$$\Phi_I(x) = \langle \vec{p}(x), [\mathbf{A}(x)]^{-1} \vec{b}_I \rangle$$

global shape functions

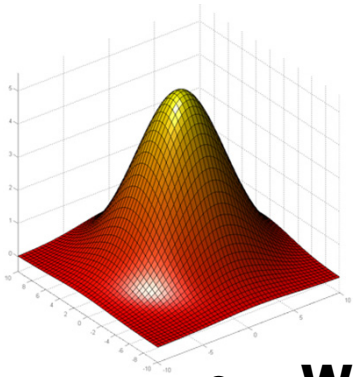
intrinsic base functions

moment matrix

$$\vec{p}^T(\vec{x}) = [1, x, y, x^2, xy, y^2, \dots, x^k, \dots, xy^{k-1}, y^k]$$

whit consistency order C_0^k

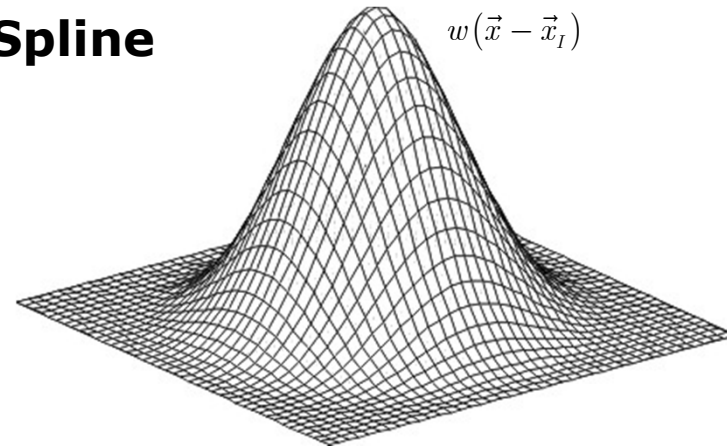




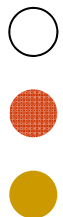
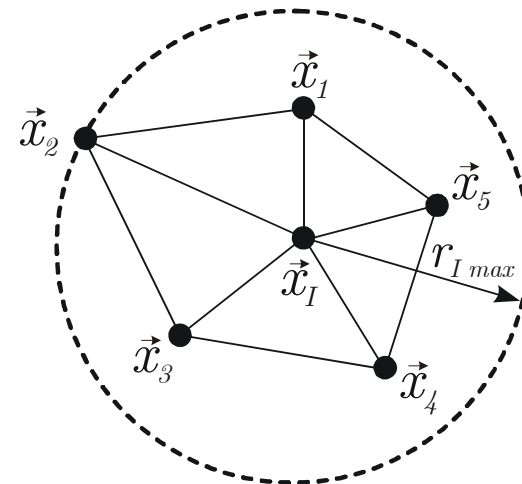
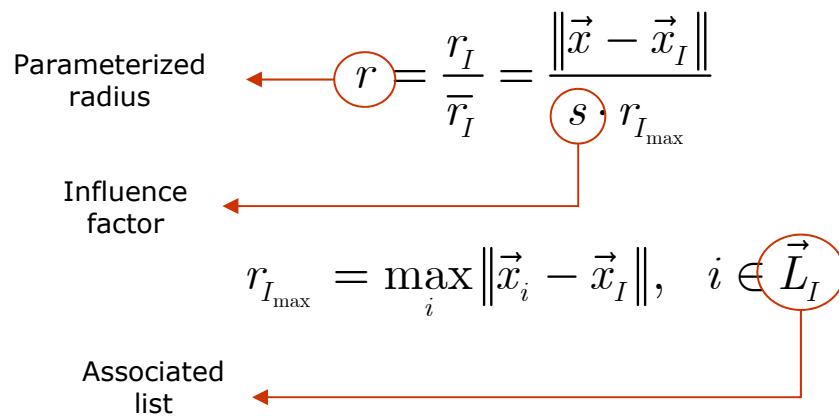
Element-Free Galerkin Method (EFG)

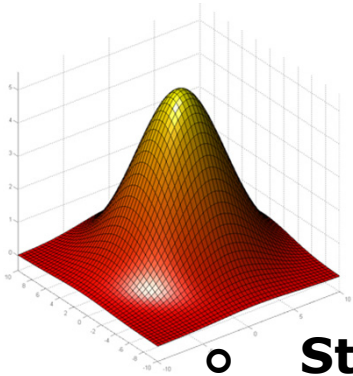
o Weight function – Quartic Spline

$$w(r) = \begin{cases} 1 - 6r^2 + 8r^3 - 3r^4 & \text{para } r \leq 1; \\ 0 & \text{para } r > 1; \end{cases}$$



o Influence Domain definition



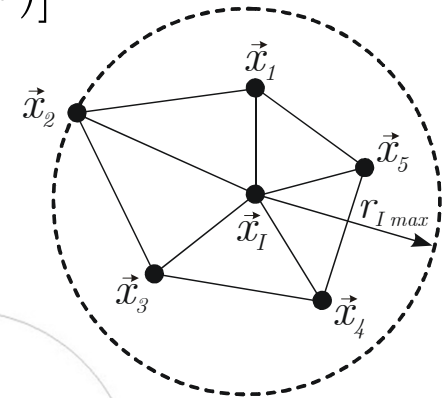


Element-Free Galerkin Method (EFG)

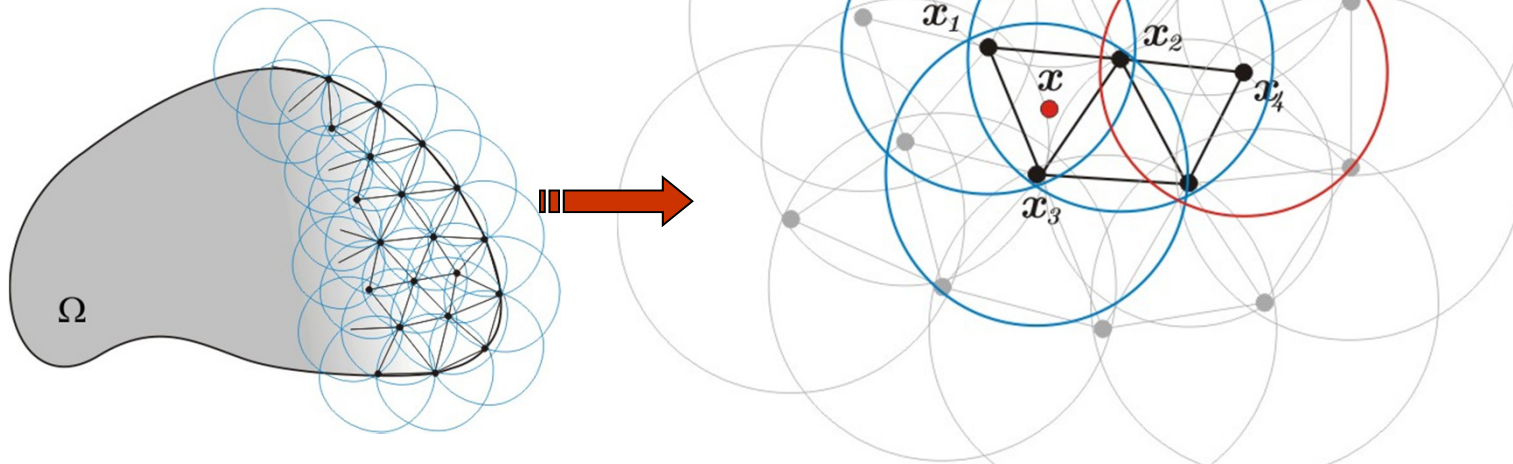
o Stability Conditions LIU *et al.* (1996)

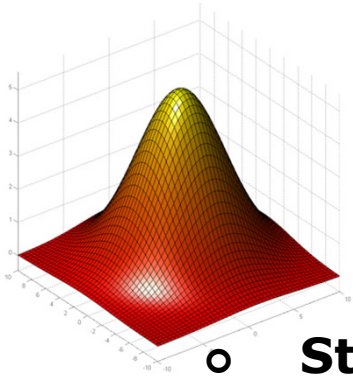
- The choice of the size of the support to assure $[\mathbf{A}(\vec{x})]^{-1}$
- The consistency order of the intrinsic base

$$\text{card} \{ \vec{x}_i | \Phi_i(\vec{x}) \neq 0 \} \geq \dim[\mathbf{A}(\vec{x})]$$



$$\vec{p}^T(\vec{x}) = [1, x, y] \quad | \quad \Omega \in \mathfrak{R}^2 \quad \forall \vec{x} \in \Omega$$

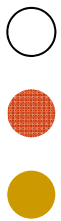
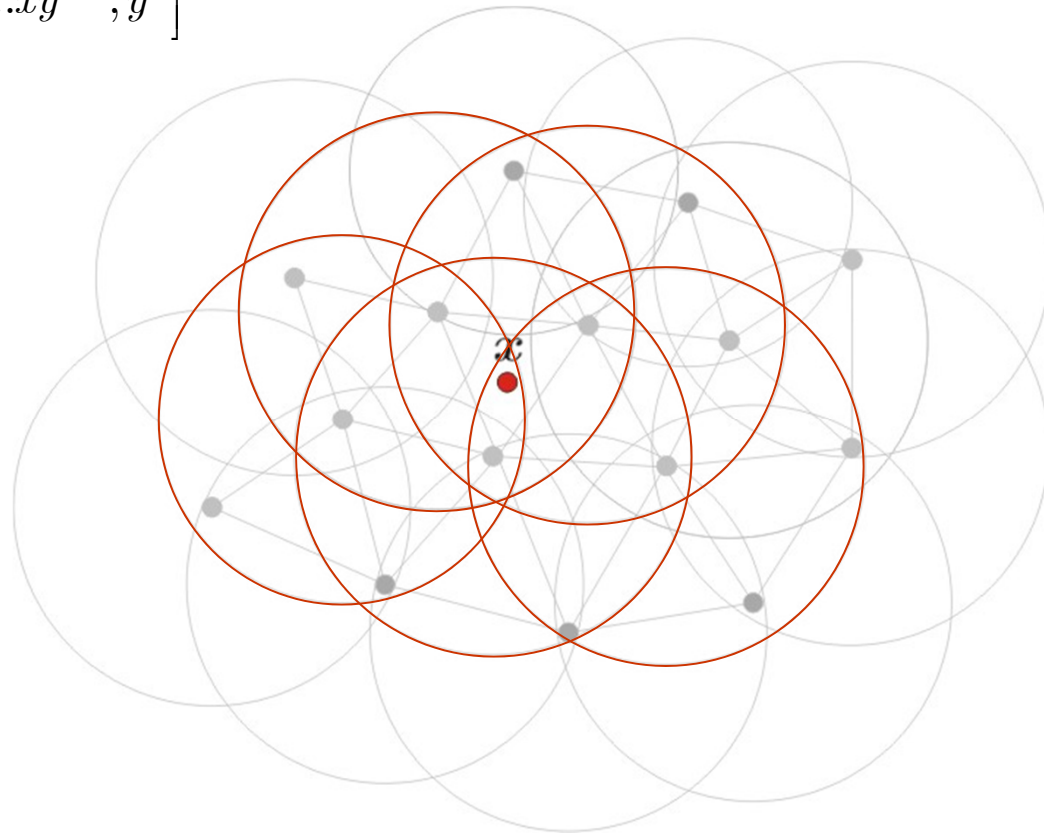
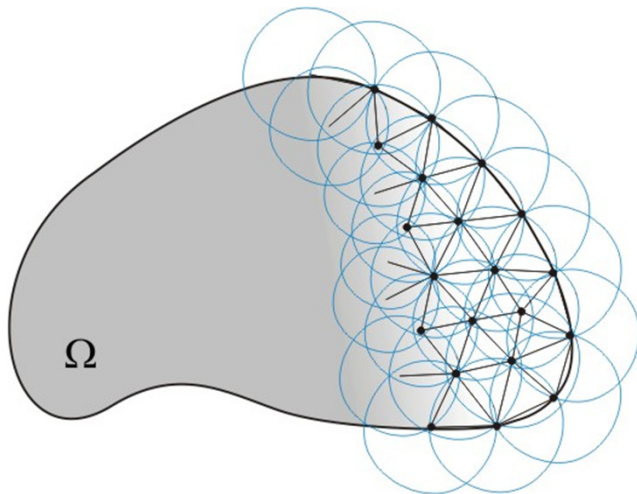


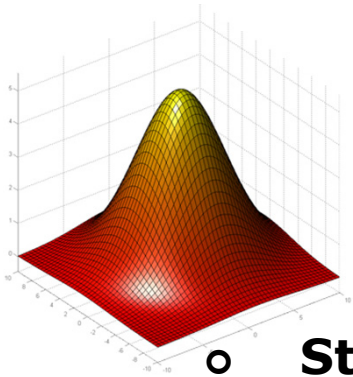


Element-Free Galerkin Method (EFG)

o **Stability Conditions** LIU *et al.* (1996)

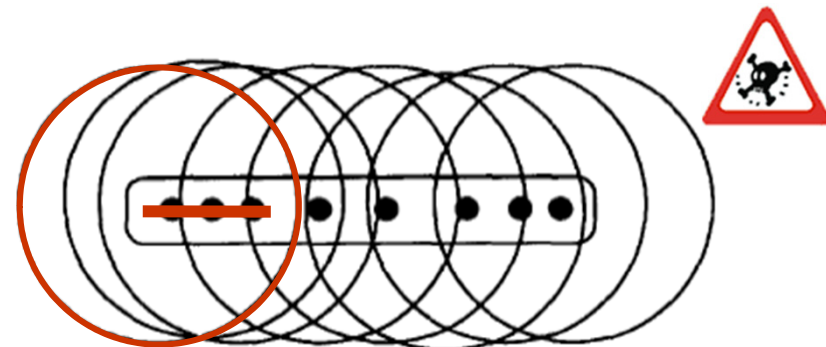
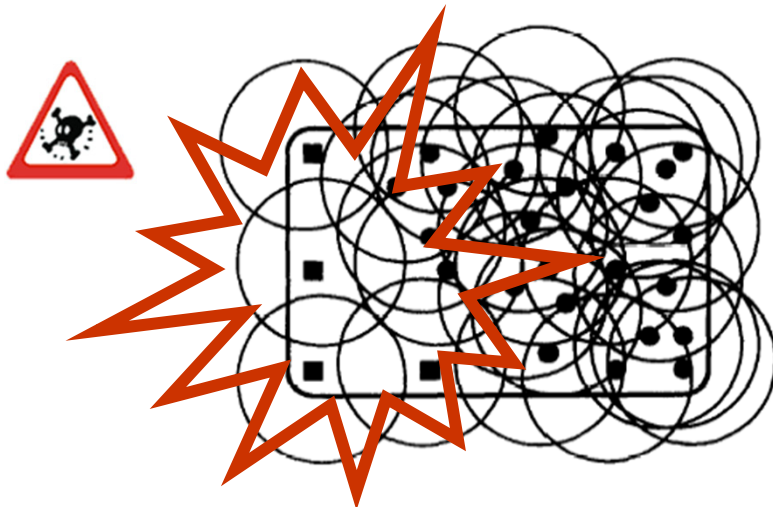
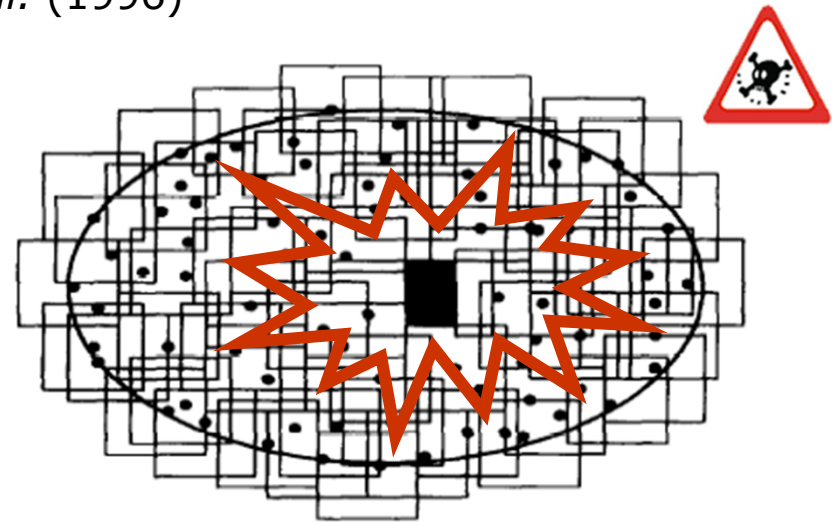
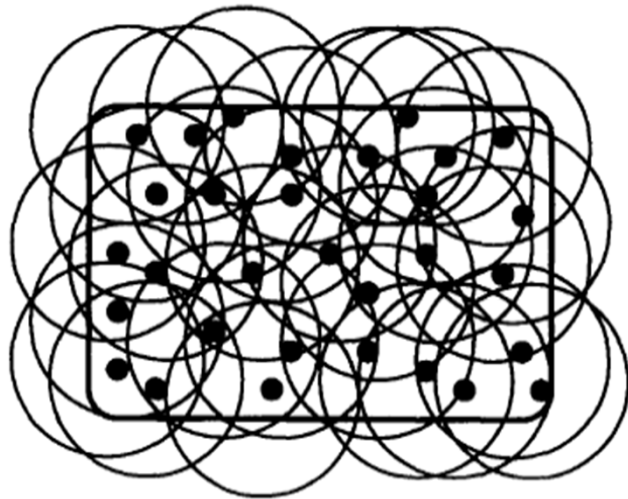
$$\vec{p}^T(\vec{x}) = [1, x, y, x^2, xy, y^2, \dots, x^k, \dots, xy^{k-1}, y^k]$$

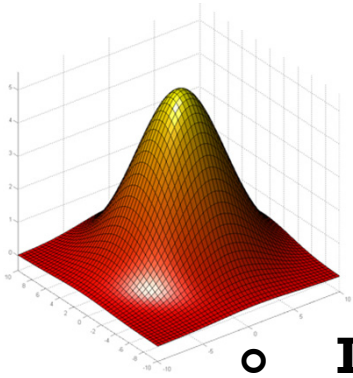




Element-Free Galerkin Method (EFG)

o **Stability Conditions** LIU *et al.* (1996)





Element-Free Galerkin Method (EFG)

- Imposition of the Essential Boundary Conditions using The Augmented Lagrange Method

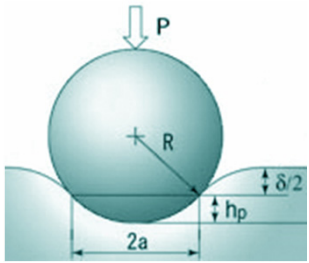
$$\Phi_I(\vec{x}_J) \neq \delta_{IJ}$$

$$\vec{Q}_{n+1}^u = - \left[\vec{\lambda}_{u_{n+1}}^k + \frac{1}{\varepsilon_u} \left(\vec{u}_{n+1}^{h^{k(i)}} - \vec{u}_{n+1} \right) \right] \quad em \Gamma_o^u$$

$$\vec{u}_{n+1}^{h^{k(i)}} = \Phi^g \vec{u}_{n+1}^{g^{k(i)}} \quad \delta \vec{u}^h = \Phi^g \delta \vec{u}^g \quad \vec{\lambda}_{u_{n+1}}^k = \mathbf{N}^g \vec{\lambda}_{u_{n+1}}^{g^k}$$

$$F^u \left(\vec{u}_{n+1}^{h^{k(i)}}, \delta \vec{u}^h \right) = - \int_{\Gamma_o^u} \vec{Q}_{n+1}^{u^{k(i)}} \cdot \delta \vec{u}^h \, d\Gamma_o^u = \vec{f}_{n+1}^{\lambda_u^k} \cdot \delta \vec{u}^g + \vec{f}_{n+1}^{u^{k(i)}} \cdot \delta \vec{u}^g$$





Unilateral Contact with Friction

○ New Problem Definition

- Find $\vec{u}_{n+1}^k \in \mathcal{K}$ that

$$\tilde{F}(\vec{u}_{n+1}^k, \delta\vec{u}) = F(\vec{u}_{n+1}^k, \delta\vec{u}) + F^u(\vec{u}_{n+1}^k, \delta\vec{u}) + F^c(\vec{u}_{n+1}^k, \delta\vec{u}) = 0 \quad \forall \delta\vec{u} \in \mathcal{V}$$

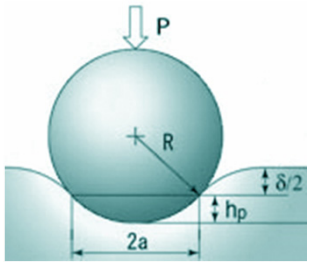
where

$$F(\vec{u}_{n+1}; \delta\vec{u}) = \int_{\Omega_o} \mathbf{P}(\vec{u}_{n+1}) \cdot \nabla_{\vec{X}} \delta\vec{u} \, d\Omega_o - \int_{\Omega_o} \rho_o \vec{b}_{n+1} \cdot \delta\vec{u} \, d\Omega_o - \int_{\Gamma_o^t} \vec{t}_{n+1} \cdot \delta\vec{u} \, dA_o$$

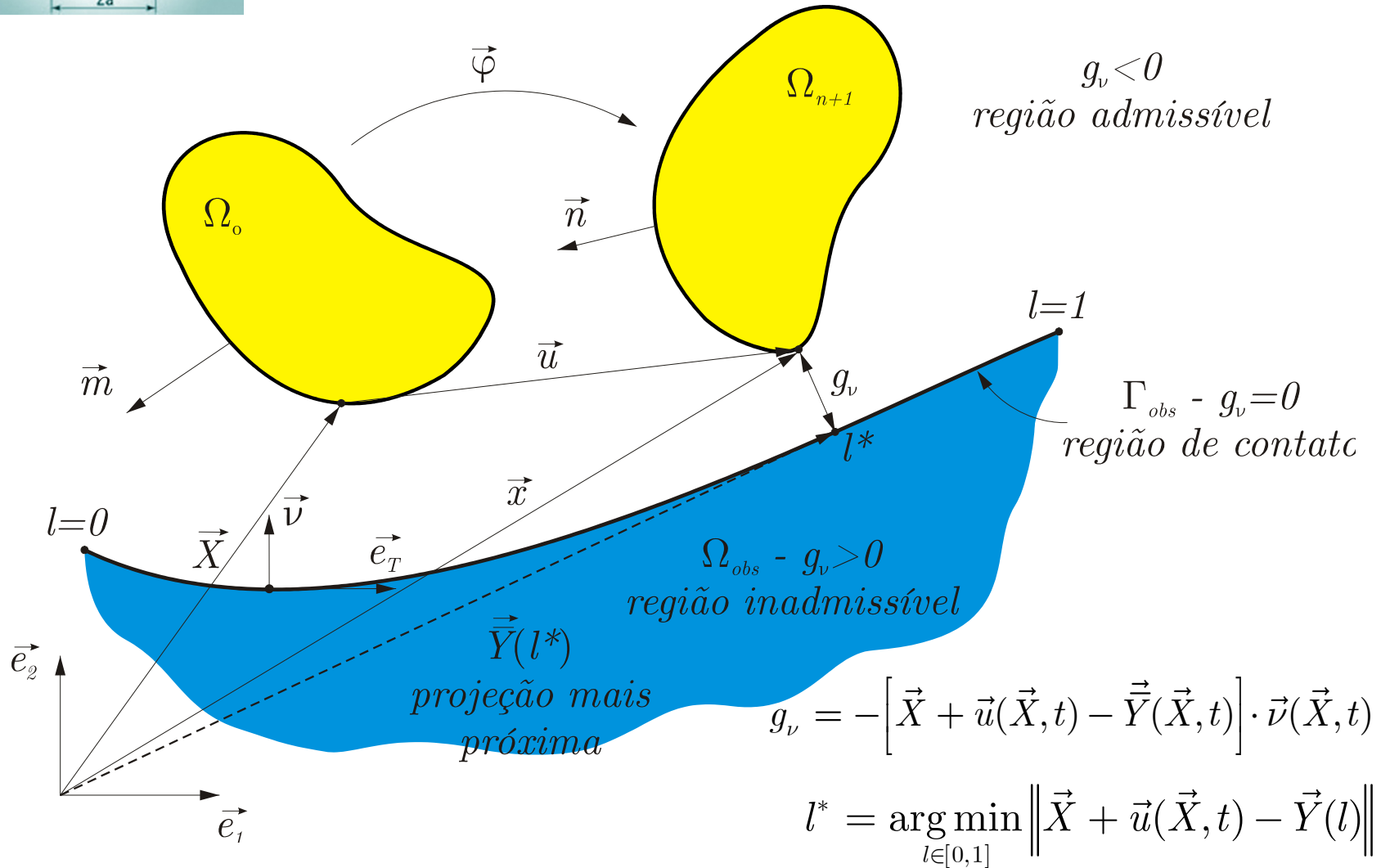
$$F^u(\vec{u}_{n+1}^k, \delta\vec{u}) = - \int_{\Gamma_o^u} \vec{Q}_{n+1}^u(\vec{u}_{n+1}^k, \varepsilon_u, \vec{\lambda}_{u_{n+1}}^k) \cdot \delta\vec{u} \, dA_o$$

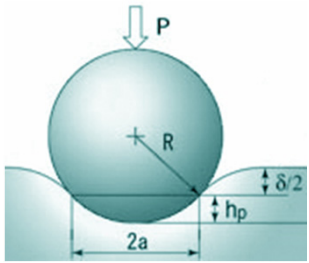
$$F^c(\vec{u}_{n+1}^k, \delta\vec{u}) = - \int_{\Gamma_o^c} \vec{Q}_{n+1}^c(\vec{u}_{n+1}^k, \varepsilon_\nu, \lambda_{\nu_{n+1}}^k) \cdot \delta\vec{u} \, dA_o$$





Unilateral Contact with Friction

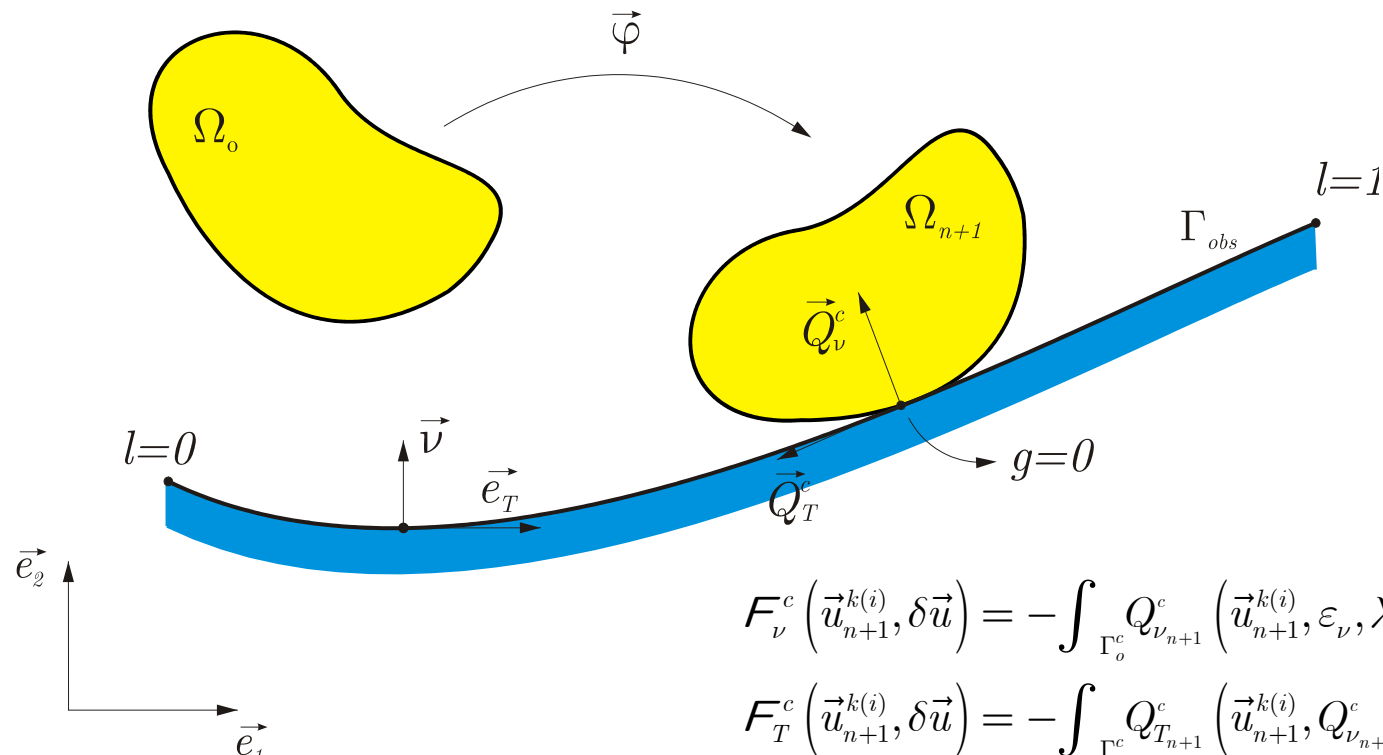




Unilateral Contact with Friction

- **Imposition of the Normal Contact and Friction Terms**
 - Normal and Tangential Works

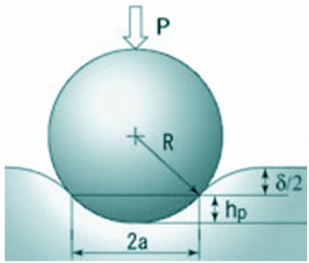
$$F^c(\vec{u}_{n+1}, \delta\vec{u}) = F_\nu^c(\vec{u}_{n+1}^{k(i)}, \delta\vec{u}) + F_T^c(\vec{u}_{n+1}^{k(i)}, \delta\vec{u})$$



$$F_\nu^c(\vec{u}_{n+1}^{k(i)}, \delta\vec{u}) = - \int_{\Gamma_o^c} Q_{\nu_{n+1}}^c(\vec{u}_{n+1}^{k(i)}, \varepsilon_\nu, \lambda_{\nu_{n+1}}^k) \vec{\nu}_{n+1}^{k(i)} \cdot \delta\vec{u} \, d\Gamma_o^c$$

$$F_T^c(\vec{u}_{n+1}^{k(i)}, \delta\vec{u}) = - \int_{\Gamma_o^c} Q_{T_{n+1}}^c(\vec{u}_{n+1}^{k(i)}, Q_{\nu_{n+1}}^c, \varepsilon_T) \vec{e}_{T_{n+1}}^{k(i)} \cdot \delta\vec{u} \, d\Gamma_o^c$$





Unilateral Contact with Friction

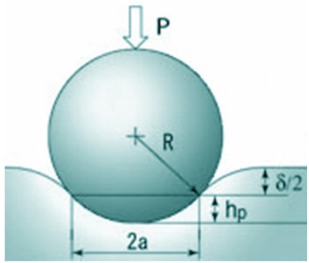
○ Normal Contact

- Augmented Lagrange Method

$$Q_{\nu_{n+1}}^c \left(\vec{u}_{n+1}^{k(i)}, \varepsilon_\nu, \lambda_{\nu_{n+1}}^k \right) = \left\langle \lambda_{\nu_{n+1}}^k + \frac{1}{\varepsilon_\nu} g \left(\vec{u}_{n+1}^{k(i)} \right) \right\rangle$$

$$F_\nu^c \left(\vec{u}_{n+1}^{h^{k(i)}}, \delta \vec{u}^h \right) = - \int_{\Gamma_o^c} Q_{\nu_{n+1}}^{k(i)} \vec{\nu}_{n+1}^{k(i)} \cdot \delta \vec{u}^h \, d\Gamma_o^c = \vec{f}_{\nu_{n+1}}^{c^{k(i)}} \cdot \delta \vec{u}^g$$





Unilateral Contact with Friction

○ Tangential Contact

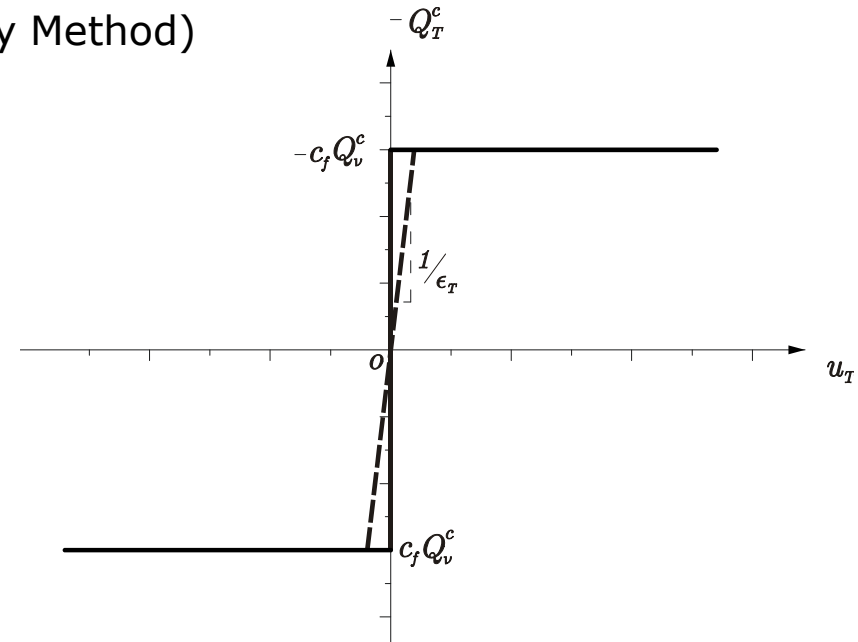
- Regularized Coulomb's Law (Penalty Method)

$$\Upsilon(Q_{T_{n+1}}^{c^{k(i)}}, Q_{\nu_{n+1}}^{c^{k(i)}}) = |Q_{T_{n+1}}^{c^{k(i)}}| - c_f Q_{\nu_{n+1}}^{c^{k(i)}}$$

$$\dot{Q}_{T_{n+1}}^{c^{k(i)}} = -\frac{1}{\epsilon_T} \left(\dot{u}_{T_{n+1}}^{k(i)} + \dot{\gamma} \frac{Q_{\nu_{n+1}}^{c^{k(i)}}}{|Q_{\nu_{n+1}}^{c^{k(i)}}|} \right)$$

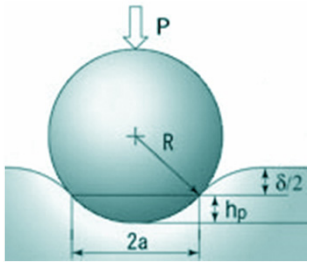
$$\dot{\gamma} \geq 0$$

$$\dot{\gamma} \Upsilon = 0$$



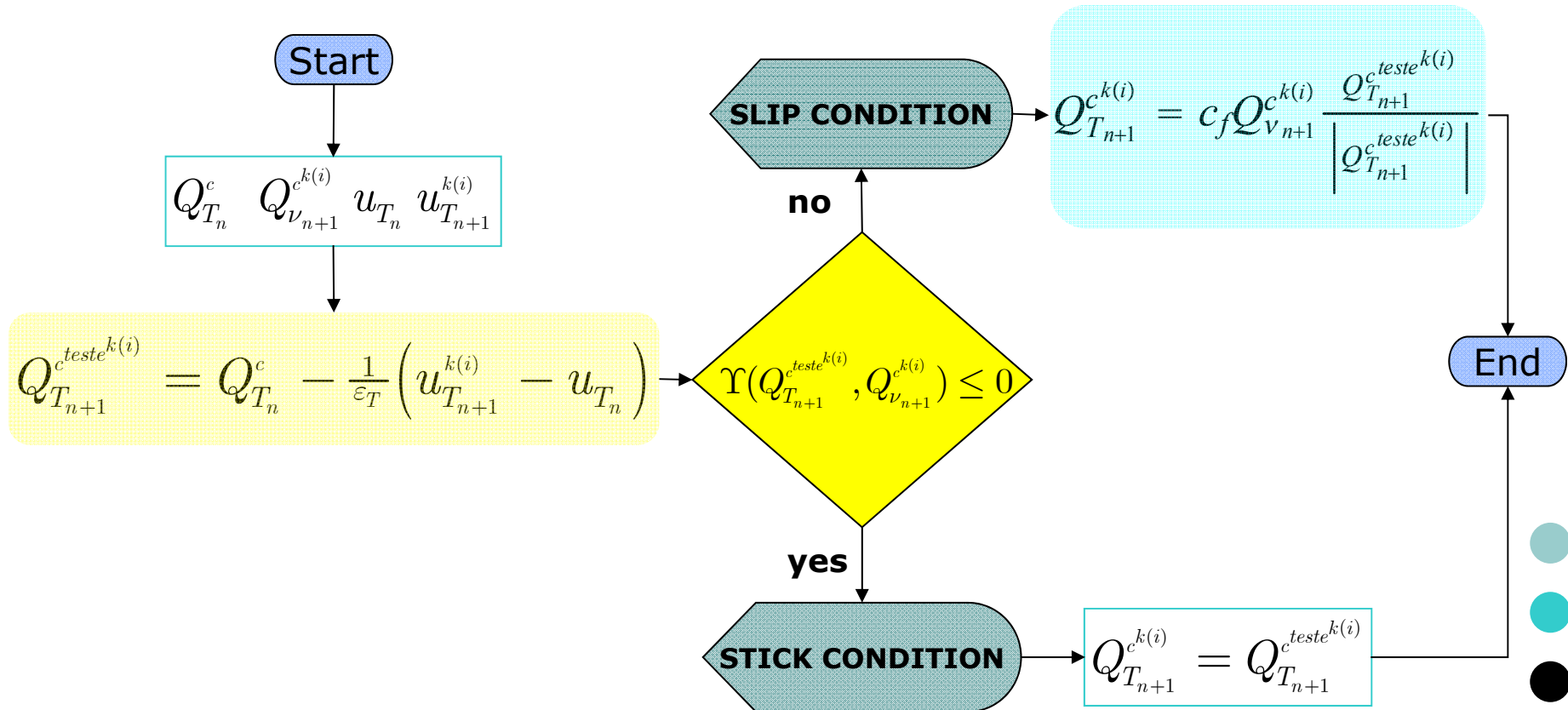
- *Stick condition* $\Upsilon \leq 0$
- *Slip condition* $\Upsilon > 0$

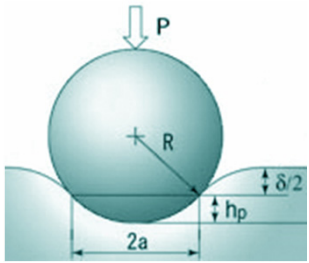




Unilateral Contact with Friction

○ Frictional Algorithm





Unilateral Contact with Friction

○ Examples using EFG

- Uniaxial Compression (with the same d.o.f of the FEM example)

$$tol_1 = 10^{-6} \quad s = 1,5 \quad 100 \text{ load steps}$$

$$\varepsilon_u = 10^{-6} \quad 7 \text{ pintg}$$

$$\tau_y^o = 0,082034 \text{ Mpa}$$

$$p_c^o = 0,040470 \text{ MPa}$$

$$\zeta = 0,50$$

$$E_m = 928,09288 \text{ MPa}$$

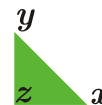
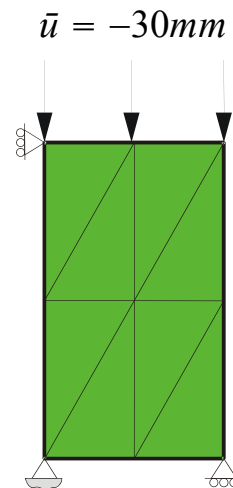
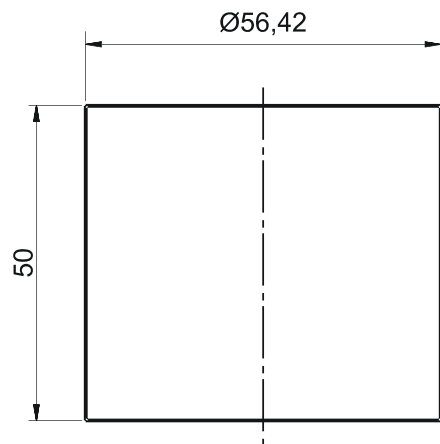
$$\rho_o^* = 0,049$$

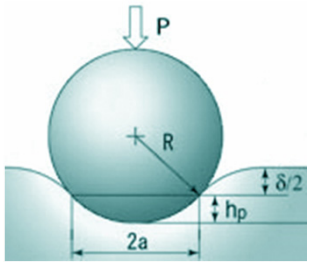
$$\nu_p = 0,00$$

$$\nu = 0,25$$

$$\gamma = 1,54$$

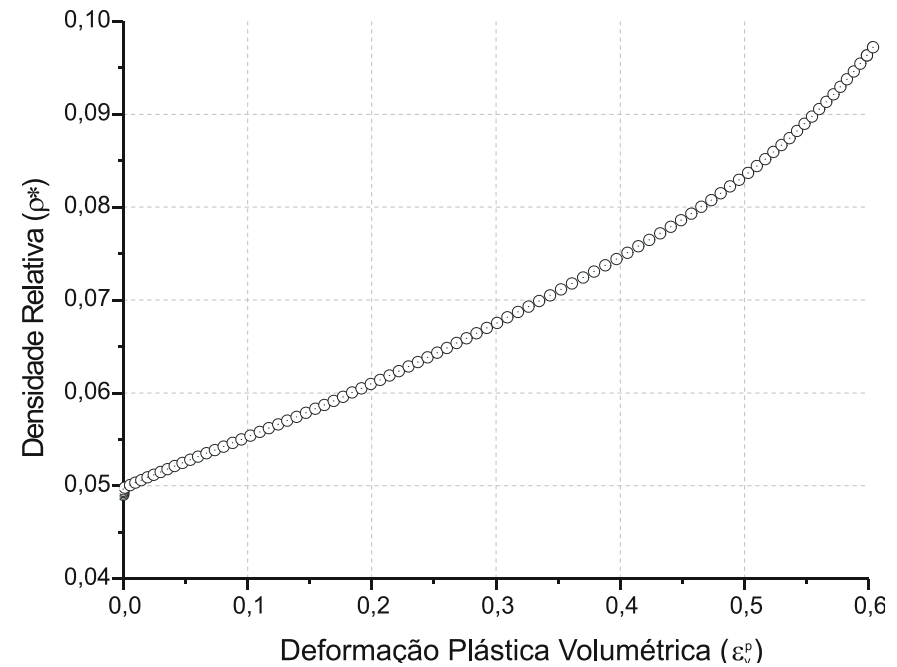
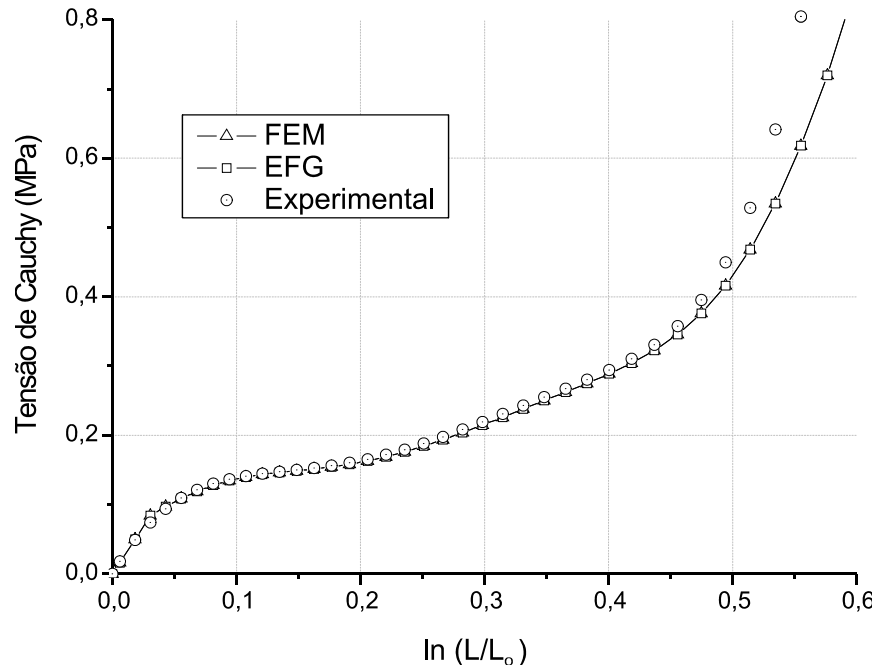
$$c = 0,30.$$





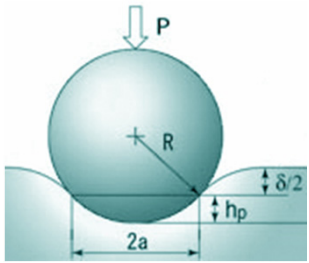
Unilateral Contact with Friction

- Uniaxial compression and the densification process



ZHANG, J. *et al* (1998)





Unilateral Contact with Friction

- Axisymmetric Cone Slice
(with the same d.o.f of the FEM example)

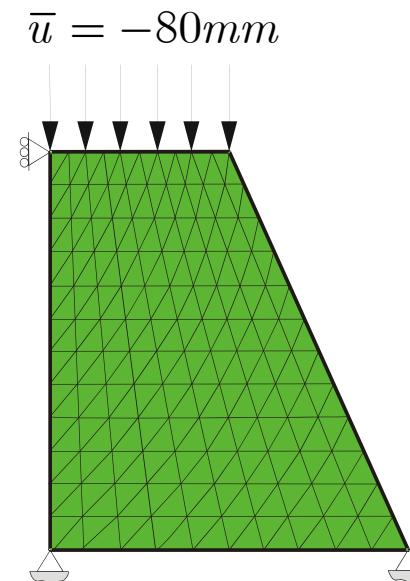
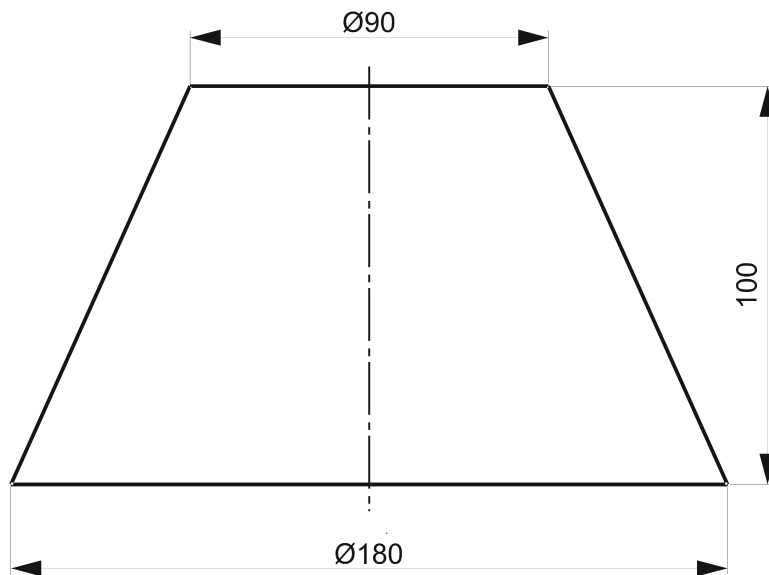
1000 load steps

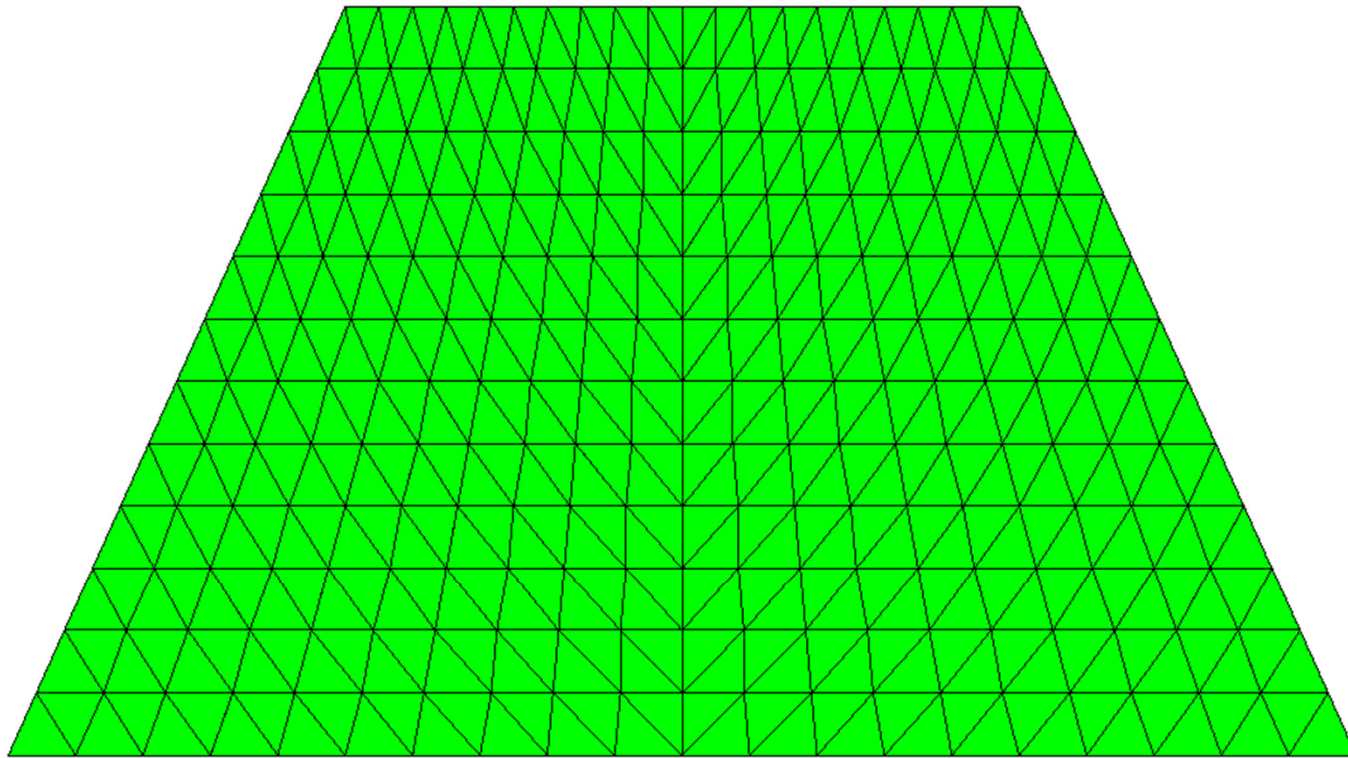
7 pintg

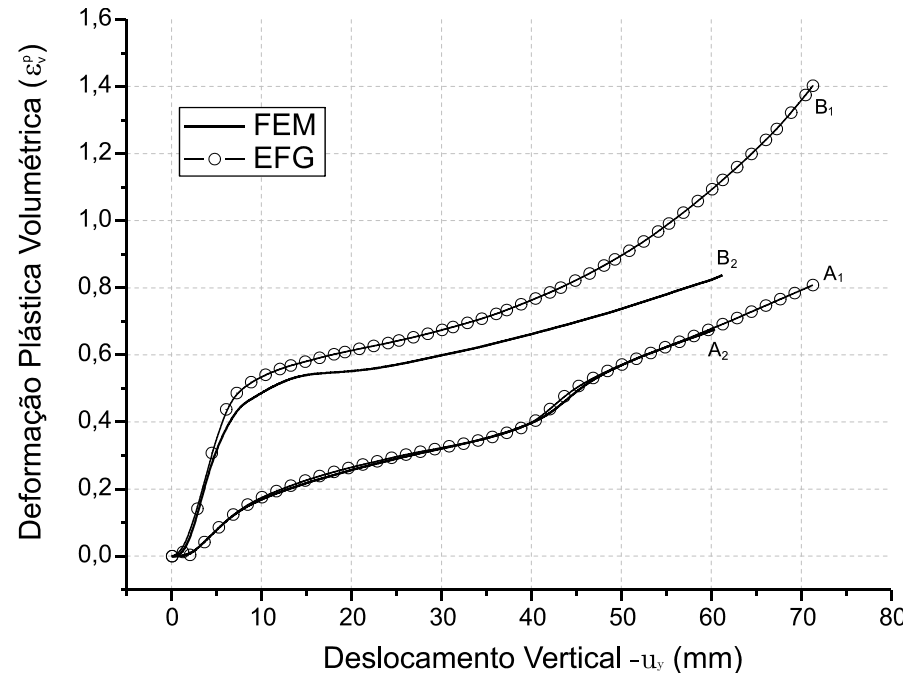
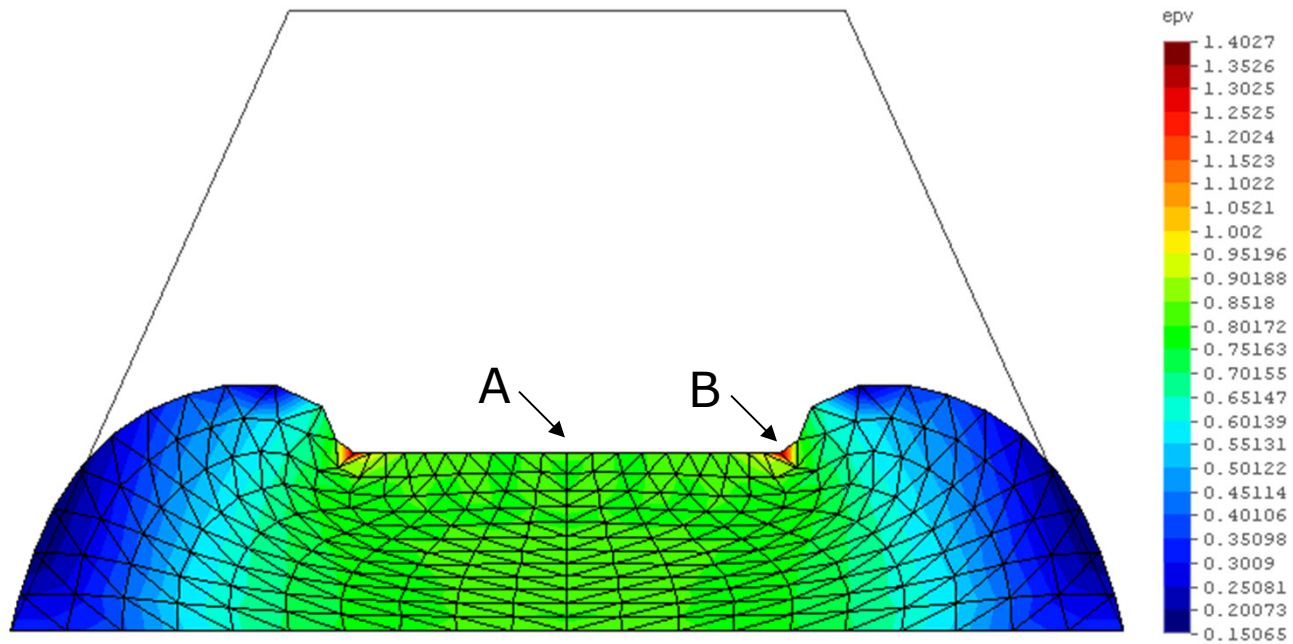
$$tol_1 = 10^{-6}$$

$$s = 1,5$$

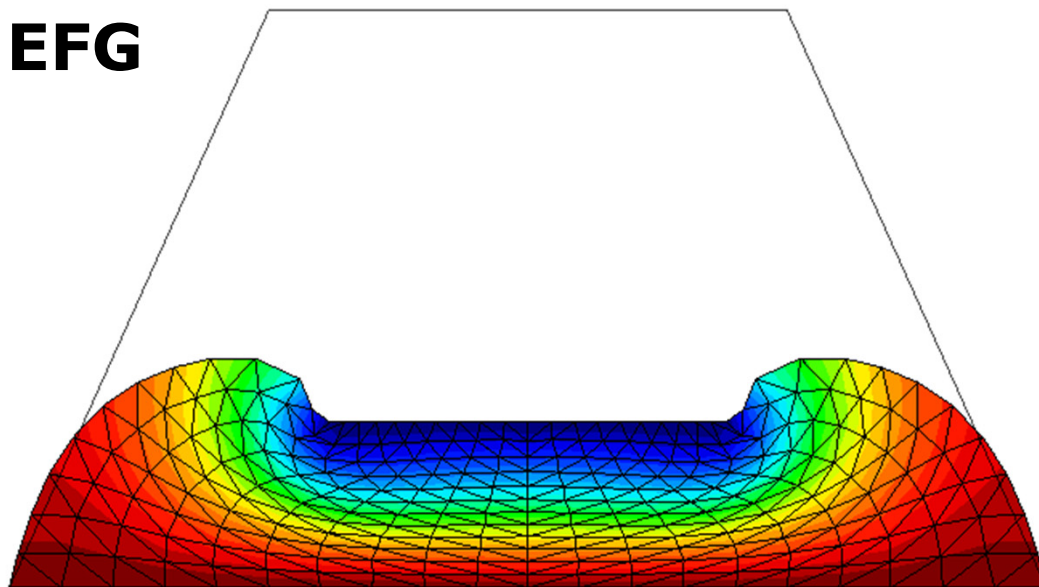
$$\epsilon_u = 10^{-6}$$





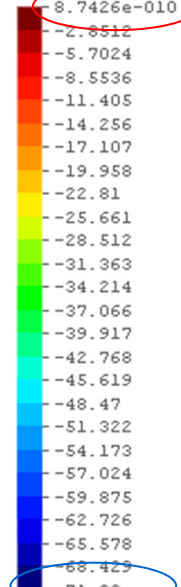


EFG



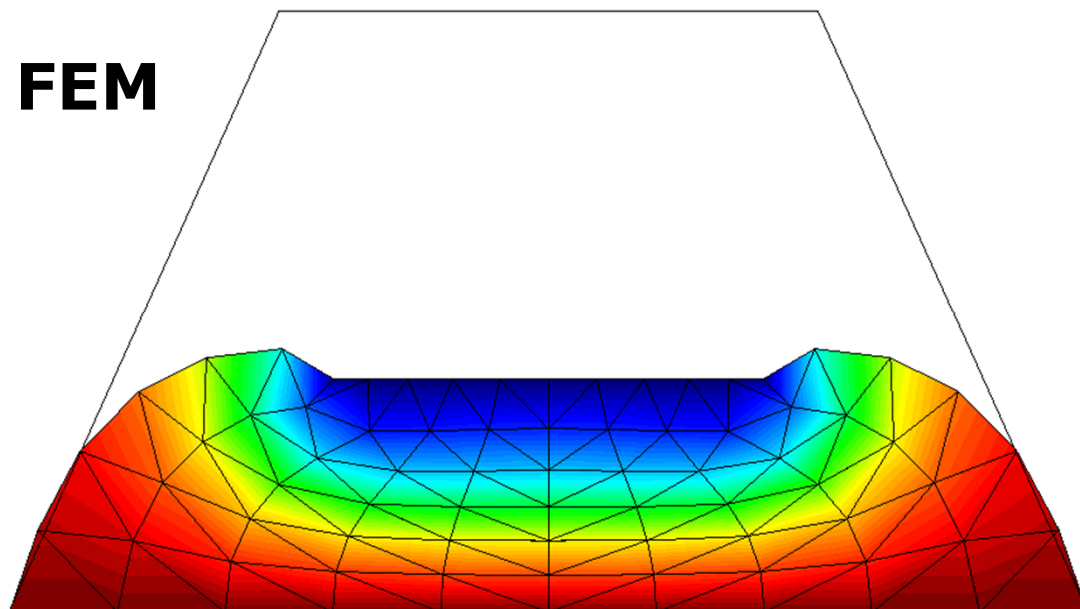
Y-DISPLACEMENT
8.7426e-010

$8,74 \cdot 10^{-10}$



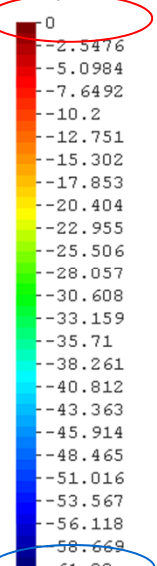
-71,28 (15%) >>

FEM

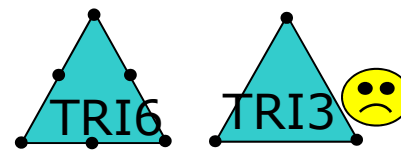


Y-Displacement

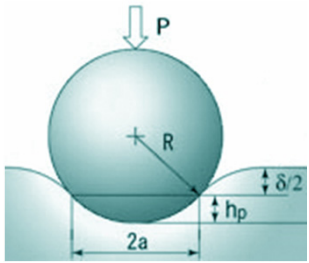
0,00



-61,22

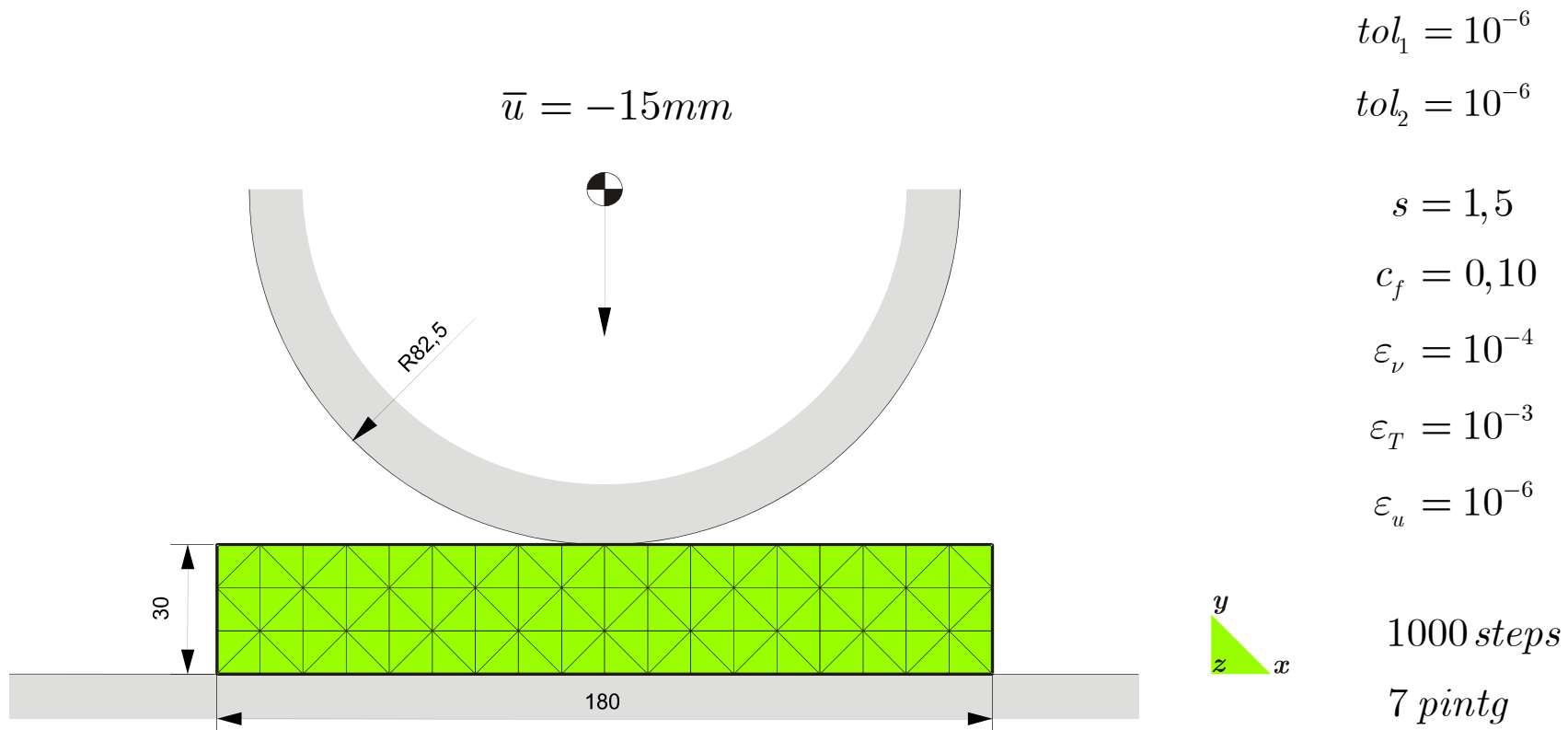


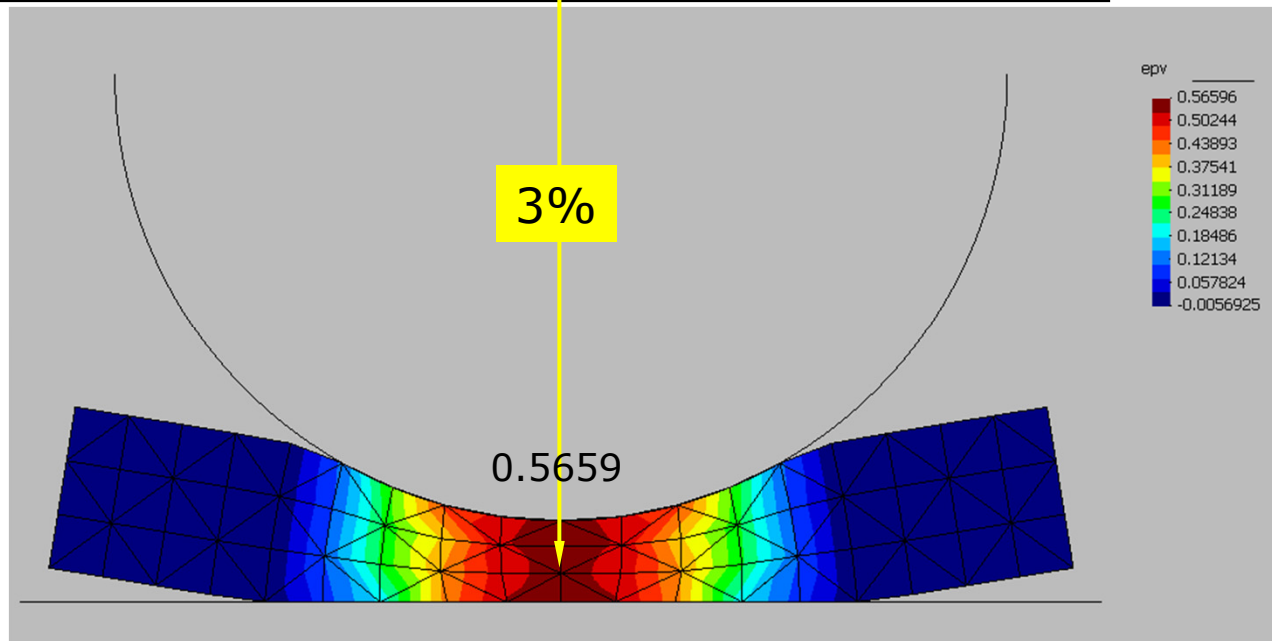
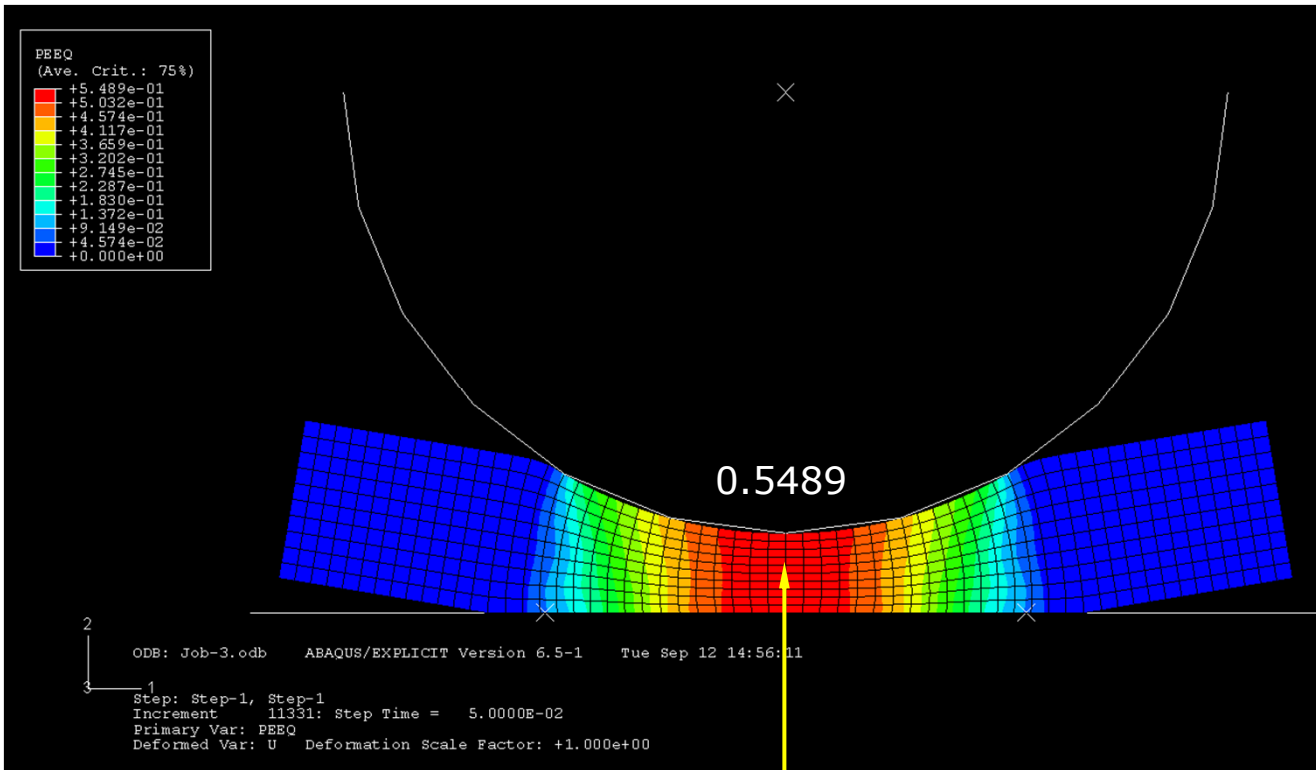
Note: Calculated with TRI6 visualized as TRI3 element!



Unilateral Contact with Friction

- Indentation Test





Conclusions

- **The constitutive model**

- The features

- a single-surface yield criteria;
- a non-associated plastic flow law;
- the relative density dependence $\mathbb{D}(\rho^*)$;

- Show a good prediction for the responses of rigid polymeric foams under simple and complex monotonic loading conditions.

- The yield surface parameters are simple and can be obtained from two independent experiments

- Uniaxial compression (ISO 844 / ASTM D1621-04a);
- Hydrostatic compression;

- The relative density dependence has as a consequence a correction in the tangent modulus.

$$\frac{\partial \bar{\boldsymbol{\tau}}}{\partial \mathbf{F}} = \left(\frac{\partial \mathbb{D}(\rho^*)}{\partial \mathbf{F}} \mathbf{E}^e \right) + \left(\mathbb{D}(\rho^*) \frac{\partial \mathbf{E}^e}{\partial \mathbf{F}} \right)$$

Conclusions

- **EFG x FEM**

- EFG method showed to be:
 - Able to withstand the analysis of very large deformation processes, without remeshing and breaking up;
 - More robust to capture high deformation levels and deformation gradients;
 - More expensive with respect the computational aspect (more integration points);
 - In another hand, load step size bigger than the FEM. (small number of interactions!)
- The choice of the penalties values (in both cases EBC and Contact) implies in changes at the convergence rate;
 - Difficulties to find the “right” value for each problem.

Conclusions

- **Numerical Aspects**

- The *Returning Map Algorithm* do not capture *critical points* (limit or path bifurcation).
 - For this reason, strategies was performed to improve the *condition number* of the local tangent modulus at the plateaus. (matrix is singular if the condition number is infinite)
- Compression causes stress bottom up due to foam consolidation.
 - Problems to insure convergence (*Turning* causes low convergence rates);
 - Further, the hardening laws are interpolated by a polynomial fitting. After 60% of the logarithm strain, this fitting don't represent the experimental data.
 - For this reason, the divergence between the numerical and experimental data is plausibly.
- At overall, the numerical procedure showed to be adequate to describes the rough non-linearities (material and contact with friction).